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# RESEARCH MEMORANDUM

APPLICATION OF SUPERSONIC VORTEX-FLOW THEORY TO THE  
DESIGN OF SUPERSONIC IMPULSE COMPRESSOR-  
OR TURBINE-BLADE SECTIONS

By Emanuel Boxer, James R. Sterrett, and John Wlodarski

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## RESEARCH MEMORANDUM

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APPLICATION OF SUPERSONIC VORTEX-FLOW THEORY TO THE  
DESIGN OF SUPERSONIC IMPULSE COMPRESSOR-  
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## SUMMARY

A method for designing shock-free supersonic impulse compressor and turbine blades in which the blade passage is essentially the space between two concentric circles is presented. Since the shock-free supersonic flow between two concentric circles is a vortex flow, the problem is one of designing an entrance to the circular-arc passage which will convert the uniform entering flow to the required vortex distribution and vice versa at the exit. The coordinates of many transition arcs have been computed and are included in tabular form. The resulting sections are all related to one another so that changes in the design variables can be investigated independently in cascade and the performance of a section for particular rotor conditions may be deduced from tests of representative sections.

Three methods of increasing the thickness, particularly near the leading and trailing edges, are presented although not experimentally investigated. The passage shape was investigated for its ability to start supersonically and the maximum design inlet Mach number for starting was determined for given vortex-blade parameters.

Cascade test results of four blade passages designed to turn the flow  $120^\circ$  at an inlet Mach number of 1.57 showed reasonable agreement with predicted surface static pressures. The stagnation pressure recovery was approximately 87.5 percent for all sections.

## INTRODUCTION

At the turn of the century, the newly realized potentialities of steam turbines led, among other things, to intensive empirical research into the proper shaping of impulse turbine blades or buckets for use

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with supersonic inlet velocities. (See reference 1.) The bucket shapes developed at that time, when little was known about supersonic flow other than the existence of the Prandtl-Meyer and Rankine-Hugoniot relations, are still the basis for design of modern impulse steam-turbine buckets since no premium has been placed upon achieving the ultimate in performance.

The present-day revolution in aircraft propulsion has brought about a demand for high-performance compressors and turbines and has instigated an intensive research in the field of high-pressure-ratio compressors and turbines with large flow-handling capacities, small frontal area, light weight, and high efficiency. With the development of the theory of the supersonic axial-flow compressor by Kantrowitz (reference 2) a new field of great promise in compressor research was opened. In particular, the entirely supersonic rotor and diffusing stator combination suggested by Kantrowitz and discussed by Ferri (reference 3) holds promise of pressure ratios per stage of 6 to 10 with efficiencies estimated from two-dimensional cascade tests to be between 70 and 80 percent. Part of the problem to be overcome for compressors of this type is the design of efficient rotor-blade sections to turn the air supersonically through large angles with very little or no reaction (that is, static-pressure rise), a requirement identical to that for efficient impulse turbine buckets. The understanding of supersonic flow has progressed rapidly in recent years. Luccini (reference 4) has demonstrated that turning passages very much more efficient than those cited by Stodola are now possible. The analytical determination of the blade shape for each design by the graphical characteristic method of solution and check testing in cascade, however, is laborious. The purpose of this paper is to present an analytical method for the design of two-dimensional related sections such that the selection of a blade for particular rotor conditions may be made quickly and easily and its performance deduced from tests of representative sections in cascade.

The principal part of the turning, in what are called vortex impulse sections, is accomplished by concentric streamlines with a vortex-type distribution of velocity for which an analytical potential-flow solution of the equation of the characteristic or Mach lines has been developed by A. Busemann. A transition section at the leading part is used to set up this vortex flow and is duplicated at the rear of the symmetrical blade to return the flow to the required uniform exit condition. The resulting sections are related to one another so that changes in the design variables, that is, design inlet and exit Mach number, blade surface Mach numbers, and turning angles can be investigated independently in cascade. Inasmuch as most practical vortex sections contract the flow, it was necessary to investigate analytically the supersonic starting problem. In addition, several methods of thickening the vanishingly thin leading and trailing edges

which are a result of the assumption of shock-free flow are suggested for practical compressor-blade application.

The effects of the boundary layer upon the potential-flow solution were obtained experimentally by several cascade tests of typical impulse blade sections at an inlet Mach number of 1.57.

#### SYMBOLS

A	area
A*	area when flow is sonic (for isentropic flow)
a	speed of sound
a*	speed of sound at point in flow for which the Mach number equals 1.0
C <sub>L</sub>	lift coefficient
C <sub>p</sub>	specific heat at constant pressure
C	reduction of maximum flow rate due to curvilinear flow
C*	nondimensional chord (chord/r*)
G*	nondimensional blade spacing (2 $\pi$ r/nr*)
K	nondimensional vortex constant
$k = \frac{VR^*}{a_0} \sqrt{\frac{\gamma - 1}{2}}$	
M	Mach number (V/a)
M*	nondimensional velocity ratio (V/a*)
m	rate of mass flow
m/r*	mass-flow parameter
n	number of blades in a rotor
p	static pressure

P	stagnation pressure
Q	vortex-flow parameter
r	radius from center of rotation of a rotor
r*	radius of sonic velocity streamline in vortex field
R	radius in vortex field
R*	nondimensional radius in vortex field ( $R/r^*$ )
s	radius ratio ( $R^*/R_1^*$ )
t	projection of added thickness normal to axial direction (fig. 9(b))
T	temperature
U	rotational velocity of rotor
u	component of velocity in x direction
v	component of velocity in y direction
V	velocity
V <sub>max</sub>	maximum velocity of flow for given stagnation conditions $\left(a_o \sqrt{\frac{\gamma - 1}{2}}\right)$
X*	nondimensional distance in x direction ( $x/r^*$ )
Y*	nondimensional distance in y direction ( $y/r^*$ )
$\beta$	inlet-flow angle, angle between relative flow and normal to rotor leading edge
$\gamma$	ratio of specific heats
$\delta$	leading-edge wedge angle
$\theta$	turning angle
$\mu$	Mach angle
$\nu$	supersonic property angle, angle through which flow must expand from $M = 1.0$ to given Mach number

$\psi$	ratio of average outlet velocity to inlet velocity
$\rho$	density
$\sigma$	solidity ( $C^*/G^*$ )
$\phi$	angle between vortex radius vector and $x$ axis, measured positive clockwise
$\phi'$	direction of flow in vortex field, measured positive clockwise from $x$ axis
$P_3/P_0$	stagnation pressure recovery
Subscripts	
$a$	axial direction
$e$	entrance condition
$i$	undisturbed inlet condition
$l$	lower or concave surface
$u$	upper or convex surface
$0$	stagnation
$s$	sonic
$2$	exit condition

A prime mark denotes conditions after normal shock.

### Blade-Section Development

To turn a gas flow through the large angles necessary in supersonic compressor- or turbine-blade passages, a vortex type of flow can be utilized if the inlet surface is properly shaped to convert the uniform inlet velocity into that corresponding to vortex flow and vice versa at the exit. The desirability of using a vortex type of flow is evident when it is realized that the maximum loading for a given peak surface pressure is achieved by uniform upper and lower surface pressures attained through the use of vortex flow. The labor involved in obtaining a solution for a blade profile is reduced because for vortex flow the equation of the Mach lines can be found and, as will be shown, only the transition

arcs need to be determined. The fact that the blades so developed are part of a related family is advantageous.

Supersonic vortex-flow theory.- As is well known, (for example, see reference 5), supersonic vortex flow is an irrotational flow the streamlines of which are concentric circles; with a constant velocity along any particular streamline. The velocity in turn varies inversely with radius.

The general vortex equation

$$VR = \text{Constant}$$

can be rewritten

$$M^*R^* = \frac{\text{Constant}}{a^*r^*} = 1.0$$

Restricting the flow to the supersonic realm will limit  $R^*$  to values

between  $\sqrt{\frac{\gamma-1}{\gamma+1}}$  and 1.0 with a flow variation shown schematically in figure 1. Since the magnitude of the velocity and its direction is known at any point, the inclination of the Mach waves through any given point may be determined as a function of  $R^*$  as shown subsequently.

Since the velocity is normal to the radius (see fig. 2), the Mach wave inclination is  $\phi \pm \mu$  or  $\phi + \frac{\pi}{2} \pm \mu$  where

$$\mu = \arcsin \frac{1}{M}$$

and

$$M = \sqrt{\frac{2M^{*2}}{(\gamma+1) - (\gamma-1)M^{*2}}}$$

so that the Mach wave inclination is  $\phi + \frac{\pi}{2} \pm \arcsin \sqrt{\frac{(\gamma+1)R^{*2} - (\gamma-1)}{2}}$

The equation of the Mach lines may be found by integration since the slope is a function of  $\phi$  and  $R^*$ . The resulting integral equation has been solved in terms of  $M^*$ . Rather than develop the direct solution,

use may be made of the fact that the Mach lines are characteristic lines and that a functional relation between velocity direction  $\phi'$  and velocity ratio  $M^*$  exists along a characteristic line (equations 81 and 89, reference 6). Since  $M^* = \frac{1}{R^*}$ , the equation of Mach lines can be written in polar form as

$$\phi = \pm \frac{1}{2} \left\{ \sqrt{\frac{\gamma + 1}{\gamma - 1}} \arcsin \left[ \frac{(\gamma - 1)}{R^{*2}} - \gamma \right] + \arcsin \left[ (\gamma + 1)R^{*2} - \gamma \right] \right\} + \text{Constant}$$

The foregoing development is based upon work done by A. Busemann (unpublished). The Mach wave network in the supersonic vortex field for  $4^\circ$  incremental changes in the value of the constant is shown in figure 3 as originally prepared by Busemann.

Generation of transition arcs.— The flow entering a blade section assumed to be uniform, supersonic, and of constant entropy must be deflected by uniquely shaped boundaries to set up the desired vortex-flow pattern. In the following exposition, it is convenient to discuss the flow in terms of flow direction,  $\phi'$ , and the property angle,  $\nu$ . Tables of functions for two-dimensional flow of a perfect gas have been published in many texts, but, for ready reference, the values of  $M$ ,  $M^*$ ,  $\mu$ , and  $R^*$  as functions of  $\nu$  for  $\gamma = 1.40$  are presented in table I.

The inlet value of  $\nu_e$  must be reduced by means of compression waves to the selected value of  $\nu_l$  on the concave surface and generally increased through expansions to the value  $\nu_u$  on the convex surface of the vortex part of the blade. Along a line where  $\phi' = 0$ , compression waves (shown in fig. 4 as solid lines) have a negative slope and expansion waves (shown dashed), a positive one. The flow (see fig. 4) must be normal to the radial line through the initial point of the most clockwise-spaced concentric arc. Since both surfaces must turn the flow an equal amount,

$$\nu_e - \nu_l + \Delta\phi = \nu_u - \nu_e$$

or

$$\Delta\phi = \nu_l + \nu_u - 2\nu_e$$



where  $\Delta\phi$  is the displacement angle between the initial points of the concentric arcs.

If the start of concave circular arc is assumed to be on the  $y^*$  axis, then the true vortex flow is bounded by the circular-arc surfaces and the expansion wave through  $\phi_1' = 0$  on the concave surface and the compression wave which passes through  $\phi_u' = \Delta\phi$  on the convex surface up to their point of intersection. Along these principal characteristic lines (heavier lines in fig. 4), the slope of the crossing characteristic or Mach wave and the flow direction is known at every point. There are expansion waves of total strength  $v_u - v_e$  crossing the principal compression characteristic and total compression wave strength of  $v_e - v_l$  crossing the principal expansion characteristic.

From the theory of characteristics as applied to supersonic flow, the direction and velocity of the flow are known to vary only across a characteristic line. When characteristics are given a finite strength, the solution of a flow problem takes the form of a network of quadrilaterals. The flow parameters within each quadrilateral are assumed constant. Thus, in the present problem, the transition arcs are generated by straight-line elements parallel to the flow within the adjacent quadrilateral or triangle formed by the principal and crossing characteristics and the transition arc itself starting from the circular arcs at  $\phi_1' = 0$  and  $\phi_u' = \Delta\phi$  and proceeding in the counter-clockwise direction until parallel with the inlet flow. Where necessary, for example, nonsymmetrical sections, the exit transition arcs at the trailing surfaces of the blade are generated in a like manner about a radius labeled  $\phi_1' = 0$  with the exception that signs of all flow angles  $\phi'$  are changed.

An illustrative example of a particular design is shown in figure 4 for an assumed inlet  $v_e$  of  $8^\circ$  and  $v_l$  and  $v_u$  equal to  $0^\circ$  and  $20^\circ$ , respectively. In each bounded region of flow there appear two numbers, the upper one of which is the flow direction  $\phi'$ , the lower one the property angle  $v$ . The strength of each wave is taken to be  $2^\circ$  so that the transition arc is composed of straight-line elements deflected  $2^\circ$  at the intersection with a characteristic line.

The coordinates of a number of transition arcs obtained algebraically for small increments of  $\phi$  are presented in table II for convex surfaces and in table III for concave surfaces. Each of the arcs originates on the  $y^*$  axis for which  $\phi' = 0$  and its length is dependent upon the inlet value of  $v_e$ . The range of values of  $v_e$ ,  $v_l$ , and  $v_u$  presented is thought to cover foreseeable applications for turbine- or compressor-blade section.

Blade-section layout.- The shape of any particular section is a function of inlet and exit Mach numbers as well as total turning angle and Mach number on the circular-arc surfaces. For a symmetrical profile, the turning angle  $\theta$  is equal to twice the air inlet angle  $\beta_e$ . The concave and convex circular arcs subtend central angles of  $\theta - 2(v_e - v_l)$  and  $\theta - 2(v_u - v_e)$ , respectively.

Once the design parameters have been selected, the flow channel can be constructed quite simply. The circular-arc radius for the selected values of  $v_l$  and  $v_u$  are obtained from table I and the coordinates of the transition arcs are computed from values given in tables II and III transformed by standard trigonometric means through an axis rotation of  $\theta/2 + (v_l - v_e)$  and  $\theta/2 + (v_e - v_u)$  degrees for the concave and convex surfaces, respectively. The convex transition arc is extended by means of a straight line parallel to the inlet or exit flow direction to the rotor leading- or trailing-edge line. To obtain the blade form, the convex surface is displaced a distance  $G^*$ , as shown in figure 5, so that the two surfaces are tangent at the leading and trailing edges. Examples of several symmetrical blade sections obtained in this manner are presented in figure 6.

Asymmetric sections may be designed similarly by treating the inlet and exit sections as separate layout problems. To obtain a sharp trailing edge, the exit area normal to the flow can be obtained from the familiar cascade relation

$$\frac{A_2}{A_e} = \frac{\cos \beta_e}{\cos(\beta_e - \theta)}$$

To satisfy the equation of continuity, the design exit value of  $v$  must correspond to that of an isentropic area change  $A_2/A_e$ .

Although a flow passage can be constructed for any selection of flow parameters, provided that the sum of transition turning on either surface does not exceed the total turning angle desired, there are solutions for which a compressor- or turbine-blade section does not exist because the surfaces are interchanged resulting in negative thickness. For most practical selections of  $v_l$ ,  $v_u$ , and  $\theta$ , however, no difficulty is encountered.

Not only the blade shape but the solidity  $\sigma$  as well is predetermined by the selection of the surface Mach numbers for a given rotor design. The solidity of a symmetrical blade can be determined analytically without the necessity of graphical construction. The necessary equations to find the solidity of a blade are given in appendix A. Since the mechanical design of a rotor due to blade-hub attachment

difficulties as well as the aerodynamic performance depend upon solidity, figures 7 and 8 have been prepared for illustrative purposes. The variation of solidity with turning angles for the case where

$v_e = \frac{1}{2}(v_u + v_l)$  and each property angle is constant in turn is presented in figure 7. Figure 8 presents results for the case where the interdependence of the property angles is removed. The lowest solidities are achieved when the divergence of surface Mach numbers or loading is greatest. For preset surface Mach numbers the solidity is affected slightly by a variation of inlet Mach number.

From the aerodynamic standpoint, low solidity is desirable because of lower total frictional losses; however, separation losses and outlet flow-divergence angles will undoubtedly be greater as the pressure rise on the convex surface at the exit transition section becomes greater at low solidities. The designer's selection of a reasonable solidity to yield the highest efficiency is facilitated by the use of vortex sections since the profiles are related to one another and therefore performance estimates may be made by interpolation of the results of cascade tests of representative sections.

#### Method of Creating Finite Leading-Edge Angle

For use in a practical compressor or turbine the supersonic vortex sections have a serious disadvantage because of the extremely thin leading edges which are subject to rapid wear due to high-velocity solid particles in the stream and to deflection due to the pressure differential existing on the two surfaces. This difficulty may be circumvented by the means outlined subsequently to create a finite wedge angle at the leading and trailing edges.

Subsonic axial velocity component.- Kantrowitz (reference 2) has shown that for subsonic axial velocities when expansion waves are generated along the entrance region (in this case the forepart of the convex surface) an oblique compression shock of strength equal to the total expansion strength will be created to obtain a steady-state condition. A small angle wedge therefore can be placed on the convex surface followed by either an expansion corner as shown in figure 9(a) or a convex arc so placed that all the expansion waves are upstream of the following blade. The expansion waves originating at B and the compression shock from A being of the same family will effectively cancel each other a short distance upstream of the cascade for axial velocities near sonic. The wave pattern in figure 9(a) composed of finite-strength expansion waves is seen to be completely cancelled within the confines of the figure for a group of blades. As in the case of an isolated airfoil in supersonic flow, the waves cancel completely only at an infinite distance from the source of disturbance so that in reality it

is only at a great distance from the cascade that the undisturbed Mach number is  $M_1$ . As the fluid approaches the cascade it oscillates with increasing amplitude about  $M_1$  as a mean. At the entrance the flow is turned and expanded to the inlet value  $M_e$  different than  $M_1$  to satisfy the equation of continuity.

The graphical procedure used to determine  $M_1$  is outlined in reference 2; however, the accuracy of the method is limited and it fails in solving the inverse problem of interest to the compressor designers, namely, that of determining  $M_e$  and  $\beta_e$  when  $M_1$  and  $\beta_1$  are given.

A simpler method capable of analytic solution is presented. When the oblique shocks are assumed to be relatively weak, the flow process may be regarded as isentropic. The last infinitesimal strength wave in the expansion fan about B to be cancelled (see fig. 9(b)) will be the Mach wave associated with the undisturbed upstream Mach number  $M_1$ . If that Mach wave is BD, then a line FD which is tangent at D to the limiting streamline AC entering the passage must be parallel to the direction  $M_1$ , and the area normal to the flow across BD is equal to  $A_1$ . The extension of line FD will pass through point E, since a line joining the points E and B is parallel and equal to  $G^*$ .

Because AC is a streamline and the flow in the expansion fan is isentropic, the change in flow direction  $\beta_1 - \beta_e$  must equal  $v_e - v_1$  to satisfy the equation of continuity.

To solve the problem analytically,  $\beta$  can be shown to be a function of Mach number since

$$\frac{A_e}{A_1} = \frac{(G^* - t) \cos \beta_e}{G^* \cos \beta_1} = \left(1 - \frac{t}{G^*}\right) \frac{\cos \beta_e}{\cos \beta_1}$$

and

$$\frac{A_e}{A_1} = \frac{A_e}{A^*} \times \frac{A^*}{A_1} = \frac{M_1^* \left( \frac{\gamma+1}{2} - \frac{\gamma-1}{2} M_1^{*2} \right)^{\frac{1}{\gamma-1}}}{M_e^* \left( \frac{\gamma+1}{2} - \frac{\gamma-1}{2} M_e^{*2} \right)^{\frac{1}{\gamma-1}}}$$

Use may be made of the values of  $v$  and  $\frac{A}{A^*}$  as a function of Mach number tabulated, for example, in reference 6 to determine the desired unknown through a trial-and-error selection of Mach numbers. Since the

solution is a function of  $t/G^*$ , the wedge angle is not uniquely determined. Care should be exercised in selecting a suitable angle such that the shock is attached and the position of the expansion corner permits the last wave of the fan to precede the following blade.

Another method utilizes a shock wave originating at the leading edge of the concave surface which will fall inside the passage and therefore will not affect the flow upstream of the leading edge. The value  $v_e$  (see fig. 9(c)) can be obtained from the oblique-shock equations (reference 6) when  $v_1$  and wedge angle  $\delta$  are known. The points A and B are the start of the transition arcs of a passage designed for an inlet Mach number equivalent to  $v_e$  and for a turning angle  $\theta = 2(\beta_1 - \delta)$  for a symmetrical section. From the point A, a straight line tangent to the transition arc is projected far enough forward so that the shock wave caused by deflecting the flow at point C intersects the point B on the opposite surface. The line BD is parallel to  $M_1$  creating an exterior corner of  $\delta^\circ$  at B. The basic leading-edge shape for  $v_e$  is indicated by EAF.

From practical considerations, the wedge angle  $\delta$  should be fairly small to yield a range of rotational speeds for which the shock will be attached as well as alleviating the possible flow separation at B due to shock - boundary-layer interaction.

Supersonic axial-velocity component. - When the axial component of the relative velocity is supersonic, another method of creating a leading-edge wedge is to have the shock originate on the leading edge of the convex surface by taking care to select a wedge angle sufficiently small to insure the shock location within the passage (see fig. 9(d)). The passage in this case is designed for an inlet Mach number equivalent to  $v_e$  and a turning angle  $\theta = 2(\beta_1 + \delta)$  for a symmetrical section. The surface AC is parallel to the undisturbed inlet velocity  $M_1$  and at A the surface is turned sharply to cancel the leading-edge shock. The thinner boundary layer at A, as compared to that at point B (fig. 9(c)) and the centrifugal force will materially aid in preventing flow separation on the concave surface at A. For this reason the performance would be expected to be superior to the second subsonic axial-velocity-component case.

For the symmetrical section, the sharp corners at the trailing edge will give rise to trailing expansion waves and oblique shocks. To a first-order approximation the exit Mach number will equal the entering  $M_1$  and the turning angle  $\theta = 2\beta_1$  for a true impulse condition.

### Consideration of Starting Contraction Ratio

In the design of a passage for supersonic flow, it is important to examine any converging portion to insure that supersonic flow can be started (see reference 7). Most practical vortex sections contract the flow. In this discussion, consideration is given only to those cases when the minimum area occurs between the concentric circular arcs. For given values of  $v_l$  and  $v_u$  it is possible to determine analytically the greatest design value of  $v_e$  for which supersonic flow starts in the passage. The flow can not be assumed one dimensional as in reference 8 because of the large velocity gradient between the channel walls. Since the starting Mach number depends both upon the stagnation-pressure loss through the normal shock and the maximum mass flow through the minimum section, the reduction of mass flow due to curvilinear flow must be obtained. The maximum inlet-design Mach number will therefore be that design value of Mach number for which the maximum rate of flow can be accommodated when the normal-shock loss and the large velocity gradient in the concentric-arc passage are taken into consideration.

When a normal shock is assumed to be spanning the entrance at the instant of starting as shown in figure 10, the gain in entropy through the shock will be constant so that the flow downstream of the shock will still be irrotational. If the circular portion of the passage is sufficiently long, a vortex type of flow will be generated. The vortex equation expressed nondimensionally (developed in appendix B) is

$$\sqrt{\frac{\gamma - 1}{2}} \frac{v}{a_0} s = K = \text{Constant}$$

The immediate problem is to determine the constant  $K = K_{\max}$  for which the rate of flow is a maximum for a given radius ratio and stagnation conditions. The derivation of the necessary equation is presented in appendix A and the resulting values of  $K_{\max}$  as a function of the ratio of the concave to convex radius is given in figure 11. The reduction in maximum mass flow  $C$  compared to that in a one-dimensional passage of width equal to the distance between the two radii is presented in figure 12.

The mass flow in the passage at the instant of starting when the normal shock spans the inlet is equal to the flow rate when started. The physical area contraction as a function of inlet Mach number and  $v_u$  and  $v_l$  may readily be determined for the started condition. If equated to the maximum contraction ratio for starting, when the flow reduction factor  $C$  is taken into consideration the maximum selected value of  $v_e$ , for which the passage can be designed to start, can be

found. The derivation of the equations and method of attack are outlined in appendix C. The limiting values of inlet  $v_e$  as a function of  $v_l$  and  $v_u$  calculated as indicated in appendix C are presented in figure 13.

The limiting inlet Mach number, however, is not valid for the thickened leading-edge sections which need to be investigated individually.

### Experimental Apparatus and Procedure

The blade models are mounted between glass side walls in the test section of the  $2\frac{1}{4}$ -by 2-inch supersonic cascade tunnel (fig. 14). This tunnel is a closed-return type in which the flow is produced by a compressor previously used as a supercharger on a V-1650-7 Packard aircraft engine. The power to operate the compressor is furnished by a 300-horsepower direct-current motor. Upstream of the test section is a 24- by 24-inch settling chamber containing three sets of fine screens. Downstream of the test section is a two-dimensional diffuser of approximately  $6^\circ$  divergence angle followed by return ducting incorporating two aircraft type of water-cooled radiators to control the air temperature. An air bleed-off valve in the settling chamber is used to control the settling-chamber pressure at from 1 to approximately 2 atmospheres. All tests herein reported, however, were made with atmospheric stagnation pressure. Schlieren photographs of the flow were made with the use of the usual two parabolic mirror systems.

Two blades forming one passage were used per set. The passage contours are accurately machined to a tolerance of 0.002-inch. In order to gain structural rigidity near the leading and trailing edges, the outer surfaces were arbitrarily designed to have finite edge angles and therefore the blade profiles are not similar nor do they represent any vortex section.

A view of the test section with one sidewall removed is presented in figure 15. The asymmetric supersonic nozzle was designed for an exit Mach number of 1.57 equivalent to a  $v$  of  $14^\circ$ . A solid wood fairing on the convex side and a flexible, adjustable extension on the concave side were used to guide the flow outside of the test passage. The blade models, of  $2\frac{1}{4}$ -inch span, were mounted on dowel pins press-fitted into bushed holes in one of the glass windows. The angle of attack of the blades could be varied with respect to the nozzle by rotating the glass. In this way the Mach number entering the passage could be varied within limits imposed by the appearance of detached bow waves on the leading edges.

Four sets of blades of different solidities were chosen for testing. Figure 16 is a photograph of one of these blades showing the static-pressure orifices and scratch marks used to create Mach wave disturbances in the flow. All of the blades had a turning angle of  $120^\circ$ . Solidities and the theoretical  $v$  values for the concentric axis and entrance conditions are as follows:

Blade	$v_l$ (deg)	$v_u$ (deg)	$v_e$ (deg)	Solidity	Figure
I	8	20	14	5.69	6(b)
II	4	24	14	3.64	6(c)
III	0	28	14	2.59	6(a)
IV	0	24	13	2.70	6(e)

In order to prevent the familiar condensation shock phenomenon caused by expanding moist air, the settling-chamber temperature was maintained at a high temperature by manual regulation of the cooling water flow. The operating compressor-pressure ratio for any configuration was obtained by increasing the motor speed until all test section pressures remained unchanged with a further increase in speed. Total and static pressures were measured inside the passage just before the end of the concave blade either by single- or multiple-tube probes. Flow surveys were made at several spanwise stations for several inlet Mach numbers. Chordwise static-pressure orifices were located at the midspan of both passage surfaces. In addition, three spanwise static-pressure orifices on both surfaces were located at the chordwise position where the survey measurements were made. The inlet total pressure and temperature were measured in the settling chamber. All pressure measurements were recorded by photographing a multiple-tube mercury manometer.



## RESULTS AND DISCUSSION

The figures and tables presenting the test results for the four passages are tabulated below:

Type of data	Blade			
	I	II	III	IV
	Figure			
Static-pressure distribution	17(a)	17(b)	17(c)	17(d)
Schlieren photographs	19	20	21	22
Total-pressure recoveries	18(a)	18(b)	18(c)	18(d)
Weighted average total-pressure recoveries	Table			
	IV(a)	IV(b)	IV(c)	IV(d)

An examination of the stagnation pressure recoveries at the exit of the blade passages (fig. 18) makes it obvious that the flow is not two dimensional. The boundary layer has "piled up" in toward the center of the convex surface. As in reference 4, there is evidence of circulatory boundary-layer flow down the side wall to the convex surface and inward toward the center of the convex blade. An explanation of this phenomenon can be deduced by a consideration of the centrifugal forces and boundary-layer effects along the side wall. Out in the free stream, the centrifugal force caused by curved streamlines is just balanced by a positive pressure gradient (toward the concave surface). In the wall boundary layer, however, this pressure gradient is greater than the centrifugal force; thus the flow moves down the side walls and inward on the convex surface. For low-aspect-ratio blades such as for these tests, the side-wall boundary-layer inflow effects are felt completely across the span at the exit of the blades. Two-dimensional flow at the spanwise center of the blades could be more closely approached by using large-aspect-ratio blades to minimize the extent of this boundary-layer inflow. The size of the existing equipment, however, prevented the use of large-aspect-ratio blades. In an actual rotor, centrifugal forces also exist in the spanwise direction and therefore the results of these tests should be considered only as conservative indications

of the performance of blades in an actual rotor. The fact that the flow is three dimensional in character in these tests should be kept in mind when inspection is made of the schlieren photographs.

An interesting experiment which shows the effect of this boundary-layer inflow was conducted by placing two boundary-layer fences on the convex surface of blade II, placed  $1/4$  inch from the glass wall. Each fence consisted of a thin plate  $1/4$  inch high and extending from 10 to 100 percent of the chord. Comparing the pressure recoveries near the convex surface of this modified blade (fig. 18(e)) with the same blade (fig. 18(b)) shows that the fences have reduced boundary-layer accumulation and separation. These particular data should be considered preliminary in nature as static-pressure measurements necessary to correct for shock losses were taken only at the blade surfaces.

Blade I was tested at a slightly higher than design inlet Mach number. This higher inlet Mach number was necessary to prevent shocks in the outer channels, caused by the configuration of the tunnel, from extending ahead of the blades. The static pressures on the surfaces of the blades were higher than design. This effect may be due to the thickening of the boundary layer in this narrow channel. The pressure rise at the rear part of the passage is explained by the slight flow separation on the convex surface as seen from the schlieren photographs (fig. 19(a)), and the consequent diffuser effect due to reduced flow areas.

Static-pressure distributions on passage II are in good agreement with the predicted values except at the rear part of the blade where separation had its usual effect. A study of midspan pressure recoveries indicated a decrease in pressure recovery at the convex wall for an increase of inlet Mach number corresponding to a change of  $v_e$  of  $1^\circ$ , (fig. 18(b)).

Supersonic flow in the passage formed by blade III could not be started at the design inlet Mach number of  $v$  equal to  $14^\circ$ . The inlet Mach number had to be increased to  $v$  value of approximately  $18^\circ$ . Figure 21(a) shows a schlieren photograph of the passage in an unstarted condition. All data presented for this blade were taken at an off-design inlet Mach number. As seen from the schlieren photograph in figure 21(b), the boundary layer separated from the convex blade surface at approximately the 30-percent-chord station, but was apparently reattached a short distance downstream in a manner typical of laminar boundary layer. The usual three-dimensional-flow effects are noted from the schlieren photographs and pressure recoveries.

In figure 13, it can be seen that, for a  $v_u$  of  $28^\circ$  and a  $v_l$  of 0, the maximum design entering Mach number is  $14.4^\circ$ . Blade III, therefore, should start at its design inlet Mach number of  $v$  equal to  $14^\circ$ .

In the derivations of the equations used to plot figure 13, however, the effects of the boundary layer have been neglected. The presence of boundary layer or flow separation would decrease the maximum mass flow, and thus decrease the maximum design entering Mach number. Since blade III was designed with an entering Mach number very close to the limit, it is not surprising that viscosity effects prevented it from starting at a  $\nu$  of  $14^\circ$ .

The inlet Mach number for blade IV is greater than design by the equivalent of a change in  $\nu$  of  $2^\circ$ . Nosing down the blades, however, caused the strong shock in the outer bypass channel to extend ahead of the convex-surface blade. Whether this result was caused solely by the tunnel configuration or by the fact that the passage would not start at a lower Mach number was not determined. It should be noted that this blade is designed close to the limiting design maximum inlet Mach number. The static-pressure distribution is in reasonable agreement with predicted values except for the effects of boundary-layer accumulation and separation at the rear of the passage.

The variation of the mass-weighted stagnation-pressure recovery for all blades was small. (See table IV.) An approximate average recovery of 87.5 percent was obtained for all sections tested. Examination of an individual passage shows a large variation of pressure recovery at the various span positions which is caused by three-dimensional flows.

A comparison of the blade loading  $\sigma C_L$  for the four blades given indicates that the experimental loading is approximately 90 percent of the theoretical value except for blade number I in which the excellent agreement should be disregarded in view of the marked divergence between predicted and measured pressures.

Blade	$\sigma C_L$	
	Theoretical	Measured
I	1.77	1.77
II	1.68	1.50
III	1.71	1.54
IV	1.76	1.62

Schlieren photographs shown in figures 19 to 22 give an indication of the flow condition in the blade passages. It is interesting to note in figure 19(c) and figure 22 the close correlation of the theoretical vortex-wave pattern, shown by the dotted line, and the actual resulting waves. The shock waves at the leading edge and in the passage are known

to be weak. This weakness is apparent by the rapid dissipation of the downstream reflections and the "roof top" behavior of the boundary layer upon meeting the shock wave on the convex blade surface. This phenomenon is typical of a laminar boundary layer, that is, rapid thickening and separation of the boundary layer upstream of the shock and creation of expansion waves as the boundary layer is rapidly thinned behind the shock. The magnitude of the increased boundary-layer thickness, for such flows, is a function of shock strength (reference 8).

The existence of compression or expansion waves at the exit of the passage can change the boundary layer upstream in the passage, and thus to some extent affect the separated region in the passage. This phenomenon acts much the same as the shock before the head of the total-pressure probe in figure 19(a) which causes the boundary layer to thicken upstream. Although it was impossible with the existing equipment to change the downstream pressure appreciably, the downstream pressure could be varied somewhat by moving the flexible wall. An example of this result is shown in the schlieren photograph in figure 19(c) where the angle of the trailing-edge shock has been changed approximately  $10^\circ$  and the flow leaving the blade is closer to the required exit direction. No appreciable changes were noted in the pressure recovery due to changing the downstream pressure by this method.

The results of the four-blade passages followed similar trends with small variations in over-all results. In general, the discrepancies between theoretical and experimental pressure distributions may be explained by the boundary-layer inflow and flow separation. These two effects acting together caused the pressure to rise in the region of the trailing edge. The data indicate that these discrepancies could be reduced by using suitable devices to minimize the boundary-layer inflow from the side walls. Very little data are available to compare with the present results in order to determine whether the vortex-blade sections are as efficient as any other type. Some data are contained in reference 4, in which a pressure recovery of 95 percent was experienced for a  $90^\circ$  turning passage of solidity 3.12, an inlet Mach number of 1.7, and designed for a static-pressure drop across the blade row. The data are not exactly comparable because of the difference in turning angle, Reynolds number, and aspect ratio. Stodola (reference 1) presents steam-bucket performance in terms of the velocity ratio  $\psi$ , which is equal to the ratio of the average outlet velocity to the inlet velocity. The value of  $\psi$  from the present tests is approximately 92 percent; whereas the upper limit for the data in reference 1 is about 80 percent when the inlet and outlet static pressure are assumed to be equal.

Moving the concave blade of set I away from the convex blade along the chord bisector line in order to determine what effect misalignment would have on the performance reduced the midspan pressure recovery from 95 to 84 percent for the condition shown in figure 23. The reason for

this result is obviously due to the large separation and standing shocks in the passage. The results of this part of the investigation point out the desirability of aerodynamically designing the passage even though the behavior of the boundary layer modified the predicted results.

### CONCLUSIONS

A method for designing shock-free supersonic impulse compressor and turbine blades in which the blade passage is essentially the space between two concentric circles is presented. Since the shock-free supersonic flow between two concentric circles is a vortex flow, the problem is one of designing an entrance to the circular-arc passage which will convert the uniform entering flow to the required vortex distribution and vice versa at the exit. The coordinates of many transition arcs have been computed and are included in tabular form. The resulting sections are related to one another so that changes in the design variables can be investigated independently in cascade and the performance of a section for particular rotor conditions may be deduced from tests of representative sections.

Three methods of increasing the thickness particularly near the leading and trailing edges are presented although not experimentally investigated. The ability of the passage to start supersonically was investigated and the limiting design inlet Mach number for starting was determined for given surface Mach numbers.

Four different blade passages designed to turn the flow  $120^\circ$  were investigated in cascade at a Mach number of approximately 1.57. The specific conclusions resulting from such tests are as follows:

1. The chordwise static-pressure distribution agrees reasonably well with design values. The discrepancies are explained by side-wall boundary-layer accumulation and flow separation.
2. The cascade flow was not two dimensional because of the circulatory boundary-layer flow.
3. The weighted stagnation pressure recovery for the entire span is approximately 87.5 percent for all sections.
4. There is little data to indicate whether these pressure recoveries are better or worse than sections differently designed although the recoveries are superior to any of the steam turbine bucket results reported by Stodola.
5. The desirability of aerodynamically designing the passage is borne out by the results obtained when the blades are incorrectly spaced.

Future research on the vortex impulse sections to determine the performance for a range of design variables and effects of leading-edge thickness should be undertaken because of the promising results obtained.

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## APPENDIX A

## CALCULATION OF THE SOLIDITY OF A SYMMETRICAL BLADE

The solidity of a symmetrical blade can be determined analytically by using the equations developed in appendix C. Equation (C7b) in appendix C can be rewritten

$$\frac{A_e}{r^*} = \frac{A_e}{A^*} Q(R_1^* - R_u^*) \quad (A1)$$

From figure 5, it can be seen that

$$G^* = \frac{A_e/r^*}{\sin\left(\frac{\pi}{2} - \beta_e\right)} \quad (A2)$$

or for a symmetrical blade since  $\theta = 2\beta_e$

$$G^* = \frac{A_e/r^*}{\cos \theta/2} \quad (A2a)$$

Combining equations (A2a) and (A1)

$$G^* = \frac{A_e}{A^*} \frac{Q(R_1^* - R_u^*)}{\cos \theta/2} \quad (A3)$$

For a symmetrical blade, the equation of the symmetrical axis (see fig. 5) is

$$X \cotn\left[\theta/2 - (\nu_e - \nu_l)\right] + Y^* = 0 \quad (A4)$$

where the  $X^*$  and  $Y^*$  axes are as defined for tables II and III.

The perpendicular distance from this axis to the end point  $X_1^*$ ,  $Y_1^*$  of the concave surface can be shown by analytic geometry to be

$$\frac{C^*}{2} = X_1^* \cos \left[ \frac{\theta}{2} - (v_e - v_l) \right] + Y_1^* \sin \left[ \frac{\theta}{2} - (v_e - v_l) \right] \quad (A5)$$

When equations (A3) and (A5) are used, the solidity for a symmetrical blade is thus given by

$$\sigma = \frac{2 \left\{ X_1^* \cos \left[ \frac{\theta}{2} - (v_e - v_l) \right] + Y_1^* \sin \left[ \frac{\theta}{2} - (v_e - v_l) \right] \right\} \cos \frac{\theta}{2}}{\frac{A_e}{A^*} Q(R_1^* - R_u^*)} \quad (A6)$$

which can be easily solved by using values from table III, (fig. 24) and the values from the following equation which have been published in many texts (for example, reference 6).

$$\frac{A_e}{A^*} = \frac{1}{M_e} \left( \frac{\frac{\gamma - 1}{2} M_e^2 + 1}{\frac{\gamma + 1}{2}} \right)^{\frac{\gamma + 1}{2(\gamma - 1)}}$$



## APPENDIX B

CALCULATION OF THE REDUCTION OF THE MAXIMUM RATE OF  
MASS FLOW DUE TO CHANNEL-WALL CURVATURE

For most practical cases, the minimum passage width which pre-determines the maximum rate of mass flow for given stagnation conditions of pressure and temperature occurs between the concentric circular-arc portions of the throat. When only this curved region of flow is considered, the boundary layer being neglected, an expression may be derived by expressing the reduction in the maximum rate of mass flow due to channel-wall curvature as a function of radius ratio.

When the normal shock spanning the entrance to the blade passage is assumed to be of constant strength (fig. 10) the flow within the passage can be considered irrotational, with a greater entropy than that upstream of the normal shock. For the flow to be irrotational and in radial equilibrium downstream of the shock if bounded by sufficiently long concentric-circular walls, it must satisfy the vortex equation

$$VR^* = \text{Constant} \quad (B1)$$

Since the flow is adiabatic and isentropic

$$\frac{\gamma}{\gamma - 1} \frac{p}{\rho} + \frac{V^2}{2} = \frac{\gamma}{\gamma - 1} \frac{p_o'}{\rho_o'} \quad (B2)$$

and

$$\frac{p}{\rho^\gamma} = \frac{p_o'}{\rho_o'^\gamma} \quad (B3)$$

combining equations (B1), (B2), and (B3) and solving for  $\rho$  gives

$$\rho = \rho_o' \left( 1 - \frac{\gamma - 1}{2a_o^2} V^2 \right)^{\frac{1}{\gamma - 1}} = \rho_o' \left( 1 - \frac{k^2}{R^{*2}} \right)^{\frac{1}{\gamma - 1}} \quad (B4)$$

where  $k = \sqrt{\frac{\gamma - 1}{2}} \frac{VR^*}{a_o}$  and  $\rho_o'$  equals the stagnation density behind the normal shock.

The mass flow parameter  $m^*$  is therefore

$$m^* = \frac{m}{r^*} = \int_{R_u^*}^{R_1^*} \rho V dR^* = \int_{R_u^*}^{R_1^*} \rho_0' \left(1 - \frac{k^2}{R^{*2}}\right)^{\frac{1}{\gamma-1}} \sqrt{\frac{2}{\gamma-1}} a_0 \frac{k}{R^*} dR^* \quad (B5)$$

for  $\gamma = 1.4$ , equation (B5) is reduced to an analytically integrable equation such that

$$m^* = \rho_0' \sqrt{5} k a_0 \left\{ \frac{(R^{*2} - k^2)^{7/2}}{5k^2 R^{*5}} - \frac{2}{15} \frac{(R^{*2} - k^2)^{3/2}}{R^{*3}} - \frac{4}{5} \frac{(R^{*2} - k^2)^{1/2}}{R^*} - \right. \\ \left. R^* \frac{(R^{*2} - k^2)^{1/2}}{5k^2} + \log \left[ R^* + (R^{*2} - k^2)^{1/2} \right] \right\}_{R_u^*}^{R_1^*} \quad (B6)$$

For all real flows,  $k$  is less than  $R^*$ . In order to determine  $k_{max}$ , that is, that value of  $k$  which will yield the maximum rate of flow between  $R_1^*$  and  $R_u^*$ , equation (B5) is differentiated under the integral sign and equated to zero; thus,

$$\frac{dm^*}{dk} = \rho_0' \sqrt{\frac{2}{\gamma-1}} a_0 \int_{R_u^*}^{R_1^*} \left\{ -\frac{2}{\gamma-1} \frac{k^2}{R^{*3}} \left(1 - \frac{k^2}{R^{*2}}\right)^{\frac{2-\gamma}{\gamma-1}} + \right. \\ \left. \frac{1}{R^*} \left(1 - \frac{k^2}{R^{*2}}\right)^{\frac{1}{\gamma-1}} \right\} dR^* = 0$$

for  $\gamma = 1.4$

$$\frac{dm^*}{dk} = \frac{m^*}{k} - \rho_0' \sqrt{5} a_0 \left[ \frac{(R^{*2} - k^2)^{5/2}}{R^{*5}} \right]_{R_u^*}^{R_1^*} \quad (B7)$$

In order to reduce the labor involved in determining the value of  $k$  for all possible values of  $R_1^*$  and  $R_u^*$ , equation (B1) may be made nondimensional.

$$\sqrt{\frac{\gamma - 1}{2}} \frac{V}{a_0} s = K \quad (B8)$$

where

$$s = \frac{R^*}{R_1^*} \quad s_1 = 1.0 \quad s_u = \frac{R_u^*}{R_1^*} \quad K = \frac{k}{R_1^*}$$

Equation (B7) is now

$$\left\{ \frac{(s^2 - K_{\max}^2)^{7/2}}{5K_{\max} s^5} - \frac{(s^2 - K_{\max}^2)^{5/2}}{s^5} - \frac{2}{15} \frac{(s^2 - K_{\max}^2)^{3/2}}{s^3} - \right. \\ \left. \frac{4}{5} \frac{(s^2 - K_{\max}^2)^{1/2}}{s} - \frac{s(s^2 - K_{\max}^2)^{1/2}}{5K_{\max}^2} + \right. \\ \left. \log \left[ s + (s^2 - K_{\max}^2)^{1/2} \right] \right\}_{s_u}^{1.0} = 0$$

The resulting values of  $K_{\max}$  for all possible radius ratios are presented in figure 11.

Equation (B6) may be rewritten

$$m^* = \sqrt{5} \rho_0' a_0 K_{\max} \left[ \frac{(s^2 - K_{\max}^2)^{5/2}}{s^5} \right]_{s_u}^{1.0} \quad (B9)$$

The maximum one-dimensional mass flow through a passage of width equal to distance between the radii is

$$\rho^* a^* (1 - s_u) = \frac{\rho_0' a_0}{\left(1 + \frac{\gamma - 1}{2}\right)^{\frac{1}{\gamma - 1}}} \sqrt{\frac{2}{\gamma + 1}} (1 - s_u) = 0.5786 \rho_0' a_0 (1 - s_u) \quad (B10)$$

Combining equations (B9) and (B10) to determine the reduction in maximum mass flow  $C$ , due to flow curvature

$$C = 1 - 3.8643 \frac{K_{\max}}{1 - s_u} \left[ \frac{(s^2 - K_{\max}^2)^{5/2}}{s^5} \right]_{s_u}^{1.0} \quad (B11)$$

The value of  $C$  is presented in figure 12 as function of  $s_u$ .

The position of the sonic radius  $s_s$  for any given radius ratio may be found upon substitution of  $a^*$  for  $V$  in equation (B8). Thus

$$s_s = K_{\max} \sqrt{\frac{\gamma + 1}{\gamma - 1}}$$

The sonic radius is very close to the geometric mean radius such that  $s_s \approx \sqrt{s_u}$  and may be used to determine  $K_{\max}$  to a good approximation for gases whose ratios of specific heats are other than 1.4.  $K_{\max}$  determined by this method is shown dashed in figure 11, and its effect upon mass flow reduction is plotted in figure 12. The maximum error in  $K_{\max}$  which occurs at the minimum radius ratio for gases of  $\gamma = 1.3$ , 1.4, and 1.5 is  $1\frac{1}{2}$ , 2, and  $2\frac{1}{2}$  percent, respectively.

## APPENDIX C

CALCULATION OF THE MAXIMUM DESIGN ENTERING MACH  
NUMBER FROM STARTING CONSIDERATIONS

Immediately preceding the starting of a particular passage, a shock will span the inlet. After the shock has passed downstream through the minimum section the rate of mass flow is unaltered. The inlet opening for any Mach number can be obtained when the mass flow through the circular passage is known. The maximum inlet design Mach number for any particular circular passage will therefore be that maximum Mach number for which the maximum rate of flow can be accommodated when the normal-shock loss is taken into consideration.

The rate of mass flow for a developed vortex can be written as

$$\frac{\dot{m}}{r^*} = \int_{R_u^*}^{R_l^*} \rho V \, dR^* \quad (C1)$$

or

$$\frac{\dot{m}}{\rho^* a^* r^*} = \int_{R_u^*}^{R_l^*} \frac{\rho V}{\rho^* a^*} \, dR^* = \int_{R_u^*}^{R_l^*} \frac{A}{A^*} \, dR^* \quad (C2)$$

and

$$\frac{A^*}{A} = M^* \left( \frac{\gamma + 1}{2} - \frac{\gamma - 1}{2} M^{*2} \right)^{\frac{1}{\gamma - 1}}$$

and

$$R^* = \frac{1}{M^*}$$

Rewriting equation (C2) gives

$$\frac{\dot{m}}{\rho^* a^* r^*} = \int_{M_l^*}^{M_u^*} \frac{\left( \frac{\gamma + 1}{2} - \frac{\gamma - 1}{2} M^{*2} \right)^{\frac{1}{\gamma - 1}}}{M^*} \, dM^* \quad (C3)$$

When inlet and vortex flow parameters are equated

$$\frac{m}{\rho^* a^* r^*} = \frac{\rho_e V_e A_e}{\rho^* a^* r^*} = \int_{M_1^*}^{M_u^*} \frac{\left( \frac{\gamma+1}{2} - \frac{\gamma-1}{2} M^{*2} \right)^{\frac{1}{\gamma-1}}}{M^*} dM^* \quad (C4)$$

or

$$\frac{A_e}{r^*} = \frac{\rho^* a^*}{\rho_e V_e} \int_{M_1^*}^{M_u^*} \frac{\left( \frac{\gamma+1}{2} - \frac{\gamma-1}{2} M^{*2} \right)^{\frac{1}{\gamma-1}}}{M^*} dM^* \quad (C5)$$

The area contraction is

$$\frac{\frac{A_e}{r^*}}{R_1^* - R_u^*} = \frac{A_e}{r^*} \left( \frac{M_1^* M_u^*}{M_u^* - M_1^*} \right) \quad \text{--- new. 0 to form --- } R^* = \frac{1}{M^*}$$

$$= \frac{\rho^* a^*}{\rho_e V_e} \int_{M_1^*}^{M_u^*} \frac{\left( \frac{\gamma+1}{2} - \frac{\gamma-1}{2} M^{*2} \right)^{\frac{1}{\gamma-1}}}{M^*} dM^* \frac{M_1^* M_u^*}{M_u^* - M_1^*} \quad (C6)$$

Let

$$Q = \frac{M_1^* M_u^*}{M_u^* - M_1^*} \int_{M_1^*}^{M_u^*} \frac{\left( \frac{\gamma+1}{2} - \frac{\gamma-1}{2} M^{*2} \right)^{\frac{1}{\gamma-1}}}{M^*} dM^* \quad (C7)$$

which is a function of the vortex surface Mach numbers only. For  $\gamma = 1.4$

$$Q = \frac{M_1^* M_u^*}{M_u^* - M_1^*} \left[ \frac{(6 - M^{*2})^{5/2}}{280} + \frac{(6 - M^{*2})^{3/2}}{28} - \frac{(6 - M^{*2})^{1/2}}{1.555} - \right. \\ \left. (1.571 \log \frac{2.45 + (6 - M^{*2})^{1/2}}{M^*}) \right]_{M_1^*}^{M_u^*} \quad (C7a)$$

thus the area contraction is

$$\frac{\rho^* a^*}{\rho_e v_e} Q \quad (C7b)$$

The value of  $Q$  is given in figure 24 as a function of  $v_l$  and  $v_u$ .

The starting maximum contraction ratio (from reference 8) is modified by the flow reduction factor  $C$  so that starting contraction ratio

$$C.R. = \frac{P_3}{P_0} \frac{A_e}{A^*} (1 - C) \quad (C8)$$

where  $\frac{P_3}{P_0}$  is the total-pressure recovery through a normal shock and

$\frac{A_e}{A^*}$  is equal to  $\frac{\rho^* a^*}{\rho_e v_e}$  and both are a function of inlet Mach number  $M_e$ .

Equating (C7b) and (C8) to determine the maximum design value of  $M_e$  such that the contraction is equal to the starting contraction ratio gives

$$\frac{Q}{1 - C} = \frac{P_3}{P_0}$$

Since

$$\frac{P_3}{P_0} = \left( \frac{7}{6} M^2 - \frac{1}{6} \right)^{-2.5} \left( \frac{0.4 M^2 + 2}{2.4 M^2} \right)^{-3.5}$$

has been tabulated as a function of  $M$  in reference 6 the maximum design Mach number for which the section will start can readily be obtained.

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TABLE I

MACH NUMBER, MACH ANGLE AND RADIUS RATIO AS FUNCTION

OF  $\nu$  FOR  $\gamma = 1.40$ 

$\nu$	$\mu$	M	M*	R*	$\nu$	$\mu$	M	M*	R*
0.0	90.00	1.000	1.000	1.000	15.0	38.55	1.605	1.428	0.7003
.1	79.46	1.017	1.014	.9862	16.0	37.61	1.639	1.448	.6906
.2	76.75	1.027	1.023	.9775	17.0	36.72	1.673	1.467	.6817
.5	72.10	1.051	1.042	.9597	18.0	35.87	1.707	1.486	.6729
1.0	67.57	1.082	1.067	.9372	19.0	35.06	1.741	1.505	.6645
1.5	64.45	1.108	1.088	.9191	20.0	34.29	1.775	1.523	.6566
2.0	62.00	1.133	1.107	.9033	22.0	32.83	1.844	1.559	.6414
2.5	59.95	1.155	1.124	.8897	24.0	31.49	1.915	1.593	.6277
3.0	58.18	1.177	1.141	.8764	25.0	30.85	1.950	1.610	.6211
3.5	56.61	1.198	1.157	.8643	26.0	30.23	1.986	1.627	.6146
4.0	55.20	1.218	1.172	.8532	28.0	29.05	2.059	1.659	.6028
4.5	53.92	1.237	1.186	.8432	30.0	27.96	2.134	1.691	.5914
5.0	52.74	1.257	1.200	.8333	32.0	26.90	2.210	1.722	.5807
6.0	50.62	1.294	1.227	.8150	34.0	25.91	2.289	1.752	.5708
7.0	48.75	1.330	1.252	.7987	35.0	25.43	2.329	1.767	.5659
8.0	47.08	1.366	1.277	.7831	36.0	24.96	2.369	1.781	.5615
9.0	45.57	1.400	1.300	.7692	38.0	24.07	2.452	1.810	.5525
10.0	44.18	1.435	1.323	.7559	40.0	23.21	2.538	1.838	.5441
11.0	42.89	1.469	1.345	.7435	45.0	21.21	2.764	1.905	.5249
12.0	41.70	1.503	1.367	.7315	50.0	19.39	3.013	1.967	.5084
13.0	40.58	1.537	1.388	.7205	55.0	17.71	3.287	2.025	.4938
14.0	39.54	1.571	1.408	.7102	60.0	16.16	3.594	2.080	.4808



TABLE II

COORDINATES OF TRANSITION SECTIONS FOR CONVEX BLADE PASSAGE  $\gamma = 1.40$ 

$-\beta$	$v_e$	$-X^*$	$Y^*$	$-\beta$	$v_e$	$-X^*$	$Y^*$	$-\beta$	$v_e$	$-X^*$	$Y^*$
$v_u = 40^\circ$				$v_u = 36^\circ$				$v_u = 32^\circ$			
0	40	0	0.5441	0	36	0	0.5614	5	27	0.0959	0.5766
1	39	0.0186	.5439	1	35	.0191	.5612	6	26	.1140	.5749
2	38	.0370	.5434	2	34	.0380	.5607	7	25	.1318	.5728
3	37	.0549	.5426	3	33	.0565	.5599	8	24	.1492	.5705
4	36	.0724	.5416	4	32	.0746	.5588	9	23	.1664	.5680
5	35	.0894	.5402	5	31	.0924	.5574	10	22	.1833	.5651
6	34	.1061	.5386	6	30	.1097	.5558	11	21	.2000	.5621
7	33	.1226	.5368	7	29	.1268	.5538	12	20	.2163	.5587
8	32	.1388	.5346	8	28	.1436	.5516	13	19	.2325	.5552
9	31	.1546	.5323	9	27	.1601	.5491	14	18	.2484	.5513
10	30	.1701	.5297	10	26	.1764	.5464	15	17	.2640	.5473
11	29	.1855	.5268	11	25	.1924	.5435	16	16	.2795	.5432
12	28	.2007	.5238	12	24	.2082	.5403	17	15	.2945	.5389
13	27	.2156	.5205	13	23	.2237	.5368	18	14	.3093	.5342
14	26	.2304	.5169	14	22	.2391	.5331	19	13	.3239	.5293
15	25	.2450	.5132	15	21	.2542	.5292	20	12	.3382	.5242
16	24	.2593	.5092	16	20	.2690	.5251	21	11	.3522	.5190
17	23	.2735	.5050	17	19	.2838	.5207	22	10	.3658	.5137
18	22	.2876	.5005	18	18	.2983	.5162	23	9	.3789	.5082
19	21	.3014	.4959	19	17	.3126	.5114	24	8	.3915	.5027
20	20	.3151	.4911	20	16	.3267	.5064	25	7	.4035	.4973
21	19	.3286	.4860	21	15	.3406	.5012	26	6	.4146	.4920
22	18	.3419	.4808	22	14	.3542	.4958	$v_u = 30^\circ$			
23	17	.3551	.4753	23	13	.3675	.4903	0	30	0	0.5913
24	16	.3706	.4686	24	12	.3806	.4846	1	29	.0203	.5911
25	15	.3859	.4616	25	11	.3934	.4788	2	28	.0403	.5906
26	14	.3999	.4569	26	10	.4058	.4729	3	27	.0598	.5898
27	13	.4096	.4520	27	9	.4177	.4669	4	26	.0792	.5886
28	12	.4176	.4458	28	8	.4291	.4610	5	25	.0980	.5871
29	11	.4293	.4394	29	7	.4399	.4551	6	24	.1163	.5854
30	10	.4407	.4330	30	6	.4499	.4495	7	23	.1344	.5833
31	9	.4517	.4265	$v_u = 34^\circ$				8	22	.1522	.5809
32	8	.4622	.4201	0	34	0	0.5707	9	21	.1697	.5783
33	7	.4720	.4138	1	33	.0196	.5705	10	20	.1869	.5754
34	6	.4810	.4079	2	32	.0389	.5700	11	19	.2039	.5723
$v_u = 38^\circ$				3	31	.0577	.5692	12	18	.2207	.5689
0	38	0	0.5525	4	30	.0761	.5681	13	17	.2371	.5652
1	37	.0190	.5523	5	29	.0942	.5667	14	16	.2533	.5614
2	36	.0375	.5519	6	28	.1119	.5650	15	15	.2692	.5573
3	35	.0556	.5511	7	27	.1293	.5630	16	14	.2848	.5529
4	34	.0734	.5500	8	26	.1464	.5607	17	13	.3001	.5484
5	33	.0908	.5486	9	25	.1633	.5582	18	12	.3150	.5437
6	32	.1080	.5470	10	24	.1798	.5555	19	11	.3297	.5388
7	31	.1247	.5451	11	23	.1961	.5524	20	10	.3433	.5340
8	30	.1410	.5429	12	22	.2122	.5492	21	9	.3570	.5288
9	29	.1572	.5405	13	21	.2277	.5457	22	8	.3710	.5233
10	28	.1732	.5378	14	20	.2433	.5420	23	7	.3835	.5181
11	27	.1889	.5349	15	19	.2591	.5379	24	6	.3953	.5131
12	26	.2043	.5318	16	18	.2742	.5337	$v_u = 28^\circ$			
13	25	.2196	.5284	17	17	.2892	.5293	0	28	0	0.6026
14	24	.2346	.5248	18	16	.3039	.5246	1	27	.0207	.6024
15	23	.2495	.5209	19	15	.3184	.5198	2	26	.0411	.6019
16	22	.2642	.5169	20	14	.3327	.5147	3	25	.0611	.6010
17	21	.2786	.5126	21	13	.3466	.5095	4	24	.0806	.5998
18	20	.2928	.5081	22	12	.3602	.5042	5	23	.0998	.5983
19	19	.3069	.5034	23	11	.3736	.4986	6	22	.1186	.5965
20	18	.3208	.4985	24	10	.3865	.4930	7	21	.1372	.5944
21	17	.3346	.4933	25	9	.3990	.4873	8	20	.1553	.5920
22	16	.3481	.4880	26	8	.4110	.4816	9	19	.1732	.5894
23	15	.3614	.4825	27	7	.4223	.4759	10	18	.1908	.5864
24	14	.3745	.4768	28	6	.4328	.4705	11	17	.2081	.5832
25	13	.3872	.4710	$v_u = 32^\circ$				12	16	.2251	.5797
26	12	.3997	.4650	0	32	0	0.5807	13	15	.2418	.5760
27	11	.4120	.4589	1	31	.0199	.5805	14	14	.2582	.5721
28	10	.4238	.4528	2	30	.0395	.5800	15	13	.2743	.5679
29	9	.4353	.4466	3	29	.0587	.5792	16	12	.2900	.5636
30	8	.4462	.4404	4	28	.0775	.5781	17	11	.3054	.5590
31	7	.4565	.4343								
32	6	.4660	.4285								

TABLE II

COORDINATES OF TRANSITION SECTIONS FOR CONVEX BLADE PASSAGE  $\gamma = 1.40$  - Concluded

$\beta$	$v_u$	$-X^*$	$Y^*$	$\beta$	$v_u$	$-X^*$	$Y^*$	$\beta$	$v_u$	$-X^*$	$Y^*$
$v_u = 28^\circ$				$v_u = 24^\circ$				$v_u = 18^\circ$			
18	10	0.3204	0.5543	13	11	0.2513	0.6002	0	18	0	0.6729
19	9	.3349	.5495	14	10	.2680	.5962	1	17	.0232	.6727
20	8	.3489	.5445	15	9	.2842	.5920	2	16	.0460	.6721
21	7	.3622	.5395	16	8	.2998	.5877	3	15	.0683	.6711
22	6	.3746	.5347	17	7	.3147	.5832	4	14	.0901	.6698
				18	6	.3288	.5788	5	13	.1115	.6681
$v_u = 26^\circ$				$v_u = 22^\circ$				6	12	.1325	.6661
0	26	0	0.6147	0	22	0	0.6416	7	11	.1530	.6638
1	25	.0211	.6145	1	21	.0220	.6414	8	10	.1730	.6611
2	24	.0419	.6140	2	20	.0432	.6409	9	9	.1924	.6582
3	23	.0623	.6131	3	19	.0645	.6399	10	8	.2113	.6551
4	22	.0823	.6119	4	18	.0859	.6386	11	7	.2294	.6517
5	21	.1019	.6104	5	17	.1064	.6370	12	6	.2467	.6482
6	20	.1211	.6085	6	16	.1266	.6351	$v_u = 16^\circ$			
7	19	.1400	.6064	7	15	.1463	.6328	0	16	0	0.6907
8	18	.1586	.6039	8	14	.1656	.6303	1	15	.0242	.6905
9	17	.1768	.6012	9	13	.1845	.6275	2	14	.0471	.6899
10	16	.1948	.5982	10	12	.2030	.6244	3	13	.0700	.6889
11	15	.2124	.5949	11	11	.2211	.6210	4	12	.0923	.6875
12	14	.2297	.5913	12	10	.2388	.6174	5	11	.1142	.6858
13	13	.2466	.5876	13	9	.2560	.6136	6	10	.1356	.6838
14	12	.2632	.5836	14	8	.2718	.6098	7	9	.1564	.6814
15	11	.2794	.5794	15	7	.2876	.6057	8	8	.1766	.6788
16	10	.2952	.5750	16	6	.3034	.6013	9	7	.1961	.6758
17	9	.3105	.5705	$v_u = 20^\circ$				10	6	.2147	.6727
18	8	.3252	.5659	0	20	0	0.6566	$v_u = 14^\circ$			
19	7	.3393	.5612	1	19	.0226	.6564	0	12	0	0.7317
20	6	.3525	.5565	2	18	.0448	.6558	1	11	.0280	.7315
$v_u = 24^\circ$				3	17	.0666	.6549	2	10	.0498	.7309
0	24	0	0.6277	4	16	.0880	.6536	3	9	.0739	.7299
1	23	.0215	.6275	5	15	.1089	.6519	4	8	.0972	.7285
2	22	.0428	.6270	6	14	.1295	.6499	5	7	.1200	.7267
3	21	.0636	.6261	7	13	.1496	.6477	6	6	.1419	.7246
4	20	.0840	.6248	8	12	.1692	.6451	$v_u = 8^\circ$			
5	19	.1040	.6232	9	11	.1885	.6422	0	8	0	0.7833
6	18	.1237	.6213	10	10	.2072	.6391	1	7	.0269	.7831
7	17	.1430	.6191	11	9	.2255	.6357	2	6	.0531	.7824
8	16	.1619	.6167	12	8	.2431	.6321	$v_u = 4^\circ$			
9	15	.1806	.6139	13	7	.2601	.6284	0	4	0	0.8536
10	14	.1988	.6108	14	6	.2761	.6245	1	3	.0290	.8534
11	13	.2167	.6075					2	2	.0559	.8527
12	12	.2342	.6040								



TABLE III

COORDINATES OF TRANSITION SECTIONS FOR CONCAVE BLADE PASSAGE  $\gamma = 1.40$ 

$-\phi$	$v_e$	$-X^*$	$Y^*$	$-\phi$	$v_e$	$-X^*$	$Y^*$	$-\phi$	$v_e$	$-X^*$	$Y^*$
$v_1 = 0^\circ$				$v_1 = 2^\circ$				$v_1 = 6^\circ$			
0	0	0	1.0000	0	2	0	0.9035	10	16	0.3223	0.7859
0.2	0.2	.0116	1.0000	1	3	.0326	.9032	11	17	.3588	.7791
0.4	0.4	.0192	.9999	2	4	.0667	.9023	12	18	.3962	.7715
0.6	0.6	.0260	.9999	3	5	.1018	.9008	13	19	.4346	.7630
0.8	0.8	.0347	.9998	4	6	.1377	.8986	14	20	.4740	.7535
1.0	1.0	.0607	.9992	5	7	.1743	.8957	15	21	.5146	.7430
2.0	2.0	.0953	.9983	6	8	.2118	.8921	16	22	.5564	.7314
3.0	3.0	.1302	.9967	7	9	.2499	.8878	17	23	.5994	.7186
4.0	4.0	.1741	.9965	8	10	.2888	.8826	18	24	.6438	.7046
5.0	5.0	.2184	.9930	9	11	.3285	.8767	19	25	.6897	.6893
6.0	6.0	.2632	.9862	10	12	.3690	.8699	20	26	.7371	.6725
7.0	7.0	.3086	.9810	11	13	.4104	.8622	21	27	.7861	.6541
8.0	8.0	.3546	.9750	12	14	.4528	.8536	22	28	.8369	.6341
9.0	9.0	.4015	.9680	13	15	.4962	.8440	23	29	.8896	.6123
10.0	10.0	.4491	.9650	14	16	.5407	.8333	$v_1 = 8^\circ$			
11.0	11.0	.4915	.9522	15	17	.5863	.8215	0	8	0	0.7833
12.0	12.0	.5409	.9420	16	18	.6332	.8085	1	9	.0278	.7831
13.0	13.0	.5975	.9295	17	19	.6814	.7942	2	10	.0562	.7823
14.0	14.0	.6494	.9171	18	20	.7279	.7795	3	11	.0853	.7810
15.0	15.0	.7025	.9033	19	21	.7788	.7624	4	12	.1150	.7792
16.0	16.0	.7571	.8882	20	22	.8344	.7428	5	13	.1453	.7768
17.0	17.0	.8132	.8716	21	23	.8888	.7224	6	14	.1764	.7739
18.0	18.0	.8707	.8534	22	24	.9450	.7003	7	15	.2081	.7702
19.0	19.0	.9299	.8336	23	25	1.0031	.6762	8	16	.2406	.7659
20.0	20.0	.9909	.8120	24	26	1.0632	.6501	9	17	.2739	.7610
21.0	21.0	1.0540	.7884	25	27	1.1256	.6216	10	18	.3080	.7553
22.0	22.0	1.1192	.7627	26	28	1.1904	.5907	11	19	.3430	.7488
23.0	23.0	1.1865	.7348	27	29	1.2576	.5571	12	20	.3788	.7415
24.0	24.0	1.2560	.7046	$v_1 = 4^\circ$				13	21	.4157	.7333
25.0	25.0	1.3309	.6704	0	4	0	0.8536	14	22	.4537	.7242
26.0	26.0	1.4059	.6346	1	5	.0305	.8533	15	23	.4927	.7141
27.0	27.0	1.4810	.5972	2	6	.0619	.8525	16	24	.5331	.7029
28.0	28.0	1.5617	.5552	3	7	.0941	.8511	17	25	.5746	.6906
29.0	29.0	1.6457	.5095	4	8	.1272	.8491	18	26	.6173	.6771
$v_1 = 1.0^\circ$				5	9	.1610	.8464	19	27	.6617	.6622
0	1.0	0	0.9374	6	10	.1954	.8431	20	28	.7076	.6460
0.2	1.2	.00662	.9374	7	11	.2306	.8391	21	29	.7552	.6282
0.4	1.4	.01343	.9374	8	12	.2666	.8344	$v_1 = 10^\circ$			
0.6	1.6	.02036	.9373	9	13	.3033	.8289	0	10	0	0.7559
0.8	1.8	.02738	.9373	10	14	.3407	.8226	1	11	.0269	.7557
1.0	2.0	.04165	.9370	11	15	.3792	.8154	2	12	.0542	.7550
2.0	3.0	.07091	.9362	12	16	.4188	.8074	3	13	.0821	.7537
3.0	4.0	.1084	.9346	13	17	.4593	.7984	4	14	.1106	.7520
4.0	5.0	.1467	.9322	14	18	.5008	.7884	5	15	.1399	.7497
5.0	6.0	.1857	.9291	15	19	.5434	.7774	6	16	.1698	.7468
6.0	7.0	.2254	.9253	16	20	.5872	.7653	7	17	.2004	.7433
7.0	8.0	.2659	.9207	17	21	.6323	.7519	8	18	.2316	.7392
8.0	9.0	.3070	.9153	18	22	.6788	.7372	9	19	.2667	.7344
9.0	10.0	.3489	.9090	19	23	.7267	.7212	10	20	.2972	.7288
10.0	11.0	.3918	.9019	20	24	.7761	.7037	11	21	.3310	.7225
11.0	12.0	.4355	.8938	21	25	.8272	.6846	12	22	.3651	.7156
12.0	13.0	.4800	.8847	22	26	.8802	.6637	13	23	.4007	.7077
13.0	14.0	.5257	.8746	23	27	.9350	.6410	14	24	.4374	.6989
14.0	15.0	.5725	.8633	24	28	.9918	.6163	15	25	.4774	.6885
15.0	16.0	.6204	.8509	25	29	1.0507	.5894	16	26	.5165	.6777
16.0	17.0	.6697	.8373	$v_1 = 6^\circ$				17	27	.5550	.6663
17.0	18.0	.7203	.8223	0	6	0	0.8152	18	28	.5967	.6531
18.0	19.0	.7733	.8059	1	7	.0290	.8150	19	29	.6399	.6386
19.0	20.0	.8259	.7880	2	8	.0587	.8142	20	30	.6846	.6228
20.0	21.0	.8810	.7684	3	9	.0891	.8129	21	31	.7311	.6054
21.0	22.0	.9381	.7471	4	10	.1203	.8110	22	32	.7794	.5864
22.0	23.0	.9969	.7239	5	11	.1522	.8084	23	33	.8296	.5656
23.0	24.0	1.0578	.6987	6	12	.1847	.8053	24	34	.8820	.5428
24.0	25.0	1.1208	.6713	7	13	.2178	.8015	25	35	.9365	.5179
25.0	26.0	1.1860	.6416	8	14	.2518	.7970	26	36	.9934	.4908
26.0	27.0	1.2288	.6093	9	15	.2866	.7918	27	37	1.0529	.4611
27.0	28.0	1.3241	.5742					28	38	1.1152	.4286
28.0	29.0	1.3970	.5362								

TABLE III

COORDINATES OF TRANSITION SECTIONS FOR CONCAVE BLADE PASSAGE  $\gamma = 1.40$  - Concluded

$-\phi$	$v_e$	$-X^*$	$Y^*$	$-\phi$	$v_e$	$-X^*$	$Y^*$	$-\phi$	$v_e$	$-X^*$	$Y^*$
$v_1 = 12^\circ$				$v_1 = 16^\circ$				$v_1 = 22^\circ$			
0	12	0	0.7317	6	22	0.1549	0.6824	0	22	0	0.6416
1	13	.0259	.7315	7	23	.1829	.6792	1	23	.0228	.6414
2	14	.0523	.7308	8	24	.2116	.6754	2	24	.0459	.6408
3	15	.0793	.7296	9	25	.2411	.6710	3	25	.0697	.6398
4	16	.1069	.7279	10	26	.2714	.6659	4	26	.0939	.6383
5	17	.1351	.7257	11	27	.3027	.6601	5	27	.1188	.6363
6	18	.1640	.7229	12	28	.3349	.6536	6	28	.1444	.6339
7	19	.1936	.7195	13	29	.3681	.6462	7	29	.1706	.6309
8	20	.2238	.7156	14	30	.4023	.6380	8	30	.1976	.6273
9	21	.2549	.7109	15	31	.4378	.6288	9	31	.2254	.6232
10	22	.2868	.7056	16	32	.4746	.6186	10	32	.2541	.6183
11	23	.3195	.6995	17	33	.5127	.6073	11	33	.2838	.6128
12	24	.3532	.6926	18	34	.5522	.5948	12	34	.3144	.6066
13	25	.3879	.6849	19	35	.5933	.5811	13	35	.3460	.5996
14	26	.4237	.6763	20	36	.6361	.5659	14	36	.3789	.5917
15	27	.4606	.6668	21	37	.6809	.5492	15	37	.4132	.5828
16	28	.4987	.6562	22	38	.7275	.5308	16	38	.4488	.5729
17	29	.5381	.6445								
18	30	.5789	.6317								
19	31	.6212	.6175								
20	32	.6651	.6020								
21	33	.7108	.5849								
22	34	.7583	.5661								
23	35	.8077	.5457								
24	36	.8592	.5232								
25	37	.9132	.4986								
26	38	.9697	.4717								
$v_1 = 14^\circ$				$v_1 = 18^\circ$				$v_1 = 26^\circ$			
0	14	0	0.7102	0	18	0	0.6729	0	26	0	0.6147
1	15	.0252	.7100	1	19	.0239	.6727	1	27	.0218	.6145
2	16	.0509	.7093	2	20	.0481	.6721	2	28	.0441	.6139
3	17	.0771	.7082	3	21	.0729	.6710	3	29	.0668	.6130
4	18	.1038	.7065	4	22	.0984	.6694	4	30	.0901	.6115
5	19	.1312	.7044	5	23	.1243	.6674	5	31	.1141	.6096
6	20	.1592	.7017	6	24	.1509	.6648	6	32	.1388	.6073
7	21	.1879	.6984	7	25	.1783	.6617	7	33	.1642	.6044
8	22	.2174	.6945	8	26	.2064	.6580	8	34	.1903	.6009
9	23	.2476	.6900	9	27	.2352	.6537	9	35	.2172	.5969
10	24	.2787	.6848	10	28	.2649	.6487	10	36	.2451	.5922
11	25	.3106	.6789	11	29	.2955	.6430	11	37	.2741	.5868
12	26	.3435	.6722	12	30	.3271	.6366	12	38	.3041	.5807
13	27	.3774	.6647	13	31	.3597	.6294				
14	28	.4124	.6563	14	32	.3934	.6213				
15	29	.4485	.6469	15	33	.4284	.6122				
16	30	.4858	.6365	16	34	.4647	.6021				
17	31	.5245	.6251	17	35	.5022	.5910				
18	32	.5648	.6124	18	36	.5414	.5786				
19	33	.6065	.5984	19	37	.5822	.5650				
20	34	.6496	.5831	20	38	.6247	.5499				
21	35	.6946	.5663	21	39	.6682	.5337				
22	36	.7415	.5478								
23	37	.7906	.5275								
24	38	.8419	.5051								
$v_1 = 16^\circ$				$v_1 = 20^\circ$				$v_1 = 30^\circ$			
0	16	0	0.6907	0	20	0	0.6566	0	30	0	0.5913
1	17	.0246	.6905	1	21	.0233	.6564	1	31	.0210	.5911
2	18	.0495	.6898	2	22	.0470	.6558	2	32	.0423	.5905
3	19	.0749	.6887	3	23	.0712	.6547	3	33	.0642	.5895
4	20	.1009	.6871	4	24	.0960	.6532	4	34	.0868	.5881
5	21	.1276	.6850	5	25	.1214	.6512	5	35	.1098	.5863
				6	26	.1475	.6487	6	36	.1337	.5840
				7	27	.1743	.6456	7	37	.1584	.5812
				8	28	.2017	.6420	8	38	.1839	.5778
				9	29	.2300	.6378	9	39	.2094	.5740
				10	30	.2591	.6329				
				11	31	.2892	.6273				
				12	32	.3203	.6210				
				13	33	.3525	.6139				
				14	34	.3858	.6059				
				15	35	.4202	.5970				
				16	36	.4560	.5870				
				17	37	.4933	.5760				
				18	38	.5322	.5637				
$v_1 = 18^\circ$				$v_1 = 24^\circ$				$v_1 = 34^\circ$			
0	18	0	0.6147	0	24	0	0.5707	0	34	0	0.5707
1	19	.0218	.6145	1	25	.0202	.5705	1	35	.0202	.5705
2	20	.0441	.6139	2	26	.0409	.5700	2	36	.0409	.5700
3	21	.0668	.6130	3	27	.0622	.5691	3	37	.0622	.5691
4	22	.0901	.6115	4	28	.0842	.5677	4	38	.0842	.5677
5	23	.1141	.6096	5	29	.1058	.5660	5	39	.1058	.5660
6	24	.1388	.6073								
7	25	.1642	.6044								
8	26	.1903	.6009								
9	27	.2172	.5969								
10	28	.2451	.5922								
11	29	.2741	.5868								
12	30	.3041	.5807								
$v_1 = 20^\circ$				$v_1 = 26^\circ$				$v_1 = 36^\circ$			
0	20	0	0.5614	0	26	0	0.5614	0	36	0	0.5614
1	21	.0212	.5612	1	27	.0212	.5612	1	37	.0212	.5612
2	22	.0418	.5607	2	28	.0418	.5607	2	38	.0418	.5607

TABLE IV

## WEIGHTED AVERAGE PRESSURE RECOVERIES (PERCENTAGE)

(a) Blade I

Source	Pressure recovery
Entire span	88.0
1.20 inches from wall	89.3
1.00 inches from wall	94.9
0.80 inches from wall	90.6
0.60 inches from wall	85.0
0.40 inches from wall	81.4
0.20 inches from wall	84.6

(b) Blade II

Source	Pressure recovery
Entire span	87.6
1.13 inches from wall	84.8
0.93 inches from wall	90.5
0.73 inches from wall	89.5
0.53 inches from wall	81.3
0.33 inches from wall	87.4
0.13 inches from wall	90.1

(c) Blade III

Source	Pressure recovery
Entire span	87.8
1.13 inches from wall	85.8
0.93 inches from wall	85.6
0.73 inches from wall	84.6
0.53 inches from wall	87.1
0.33 inches from wall	90.8
0.13 inches from wall	91.0

(d) Blade IV

Source	Pressure recovery
Entire span	86.9
1.20 inches from wall	88.7
1.00 inches from wall	88.6
0.80 inches from wall	81.8
0.60 inches from wall	83.2
0.40 inches from wall	92.0
0.20 inches from wall	85.3



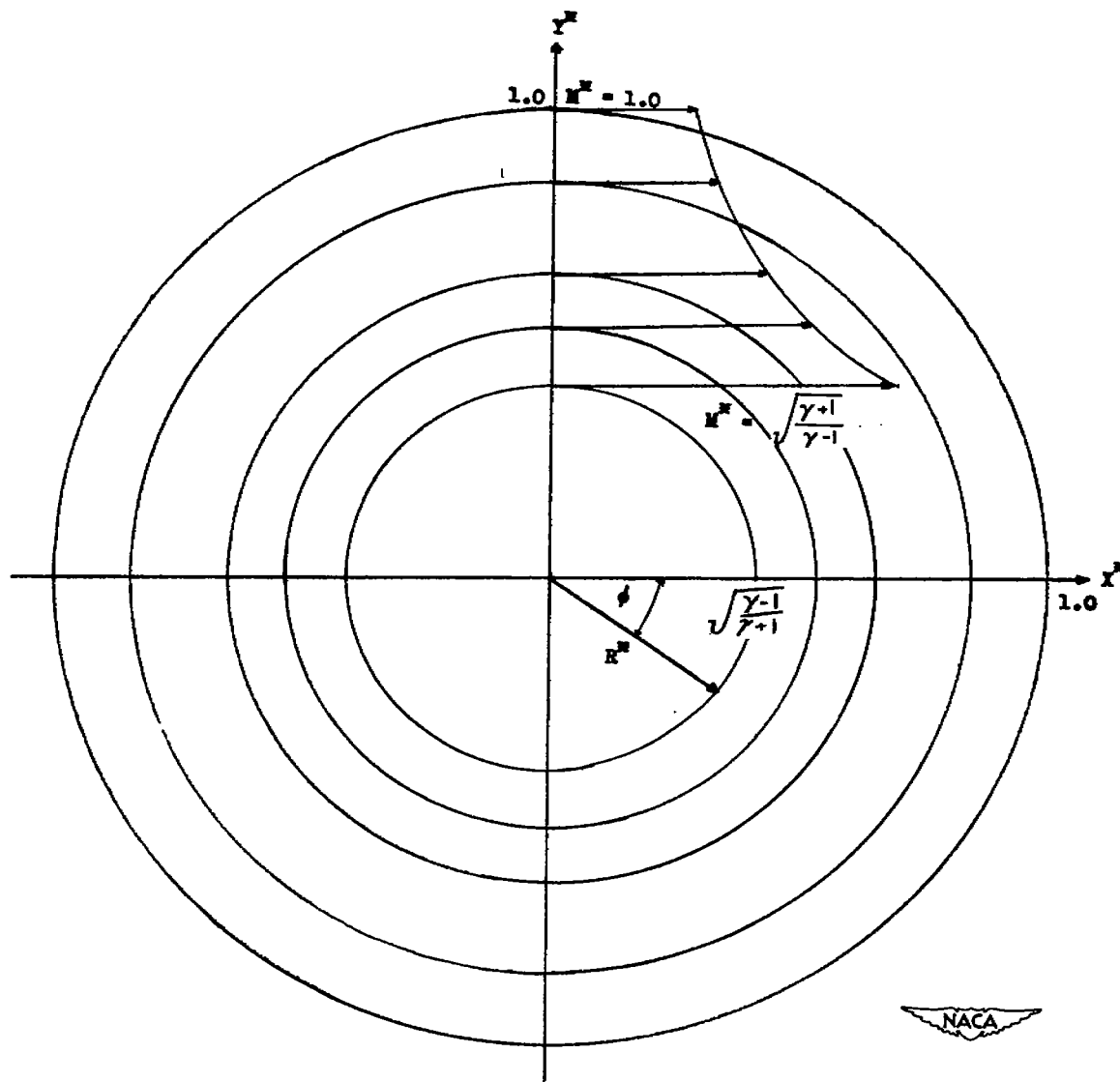


Figure 1.- Supersonic realm of vortex flow.

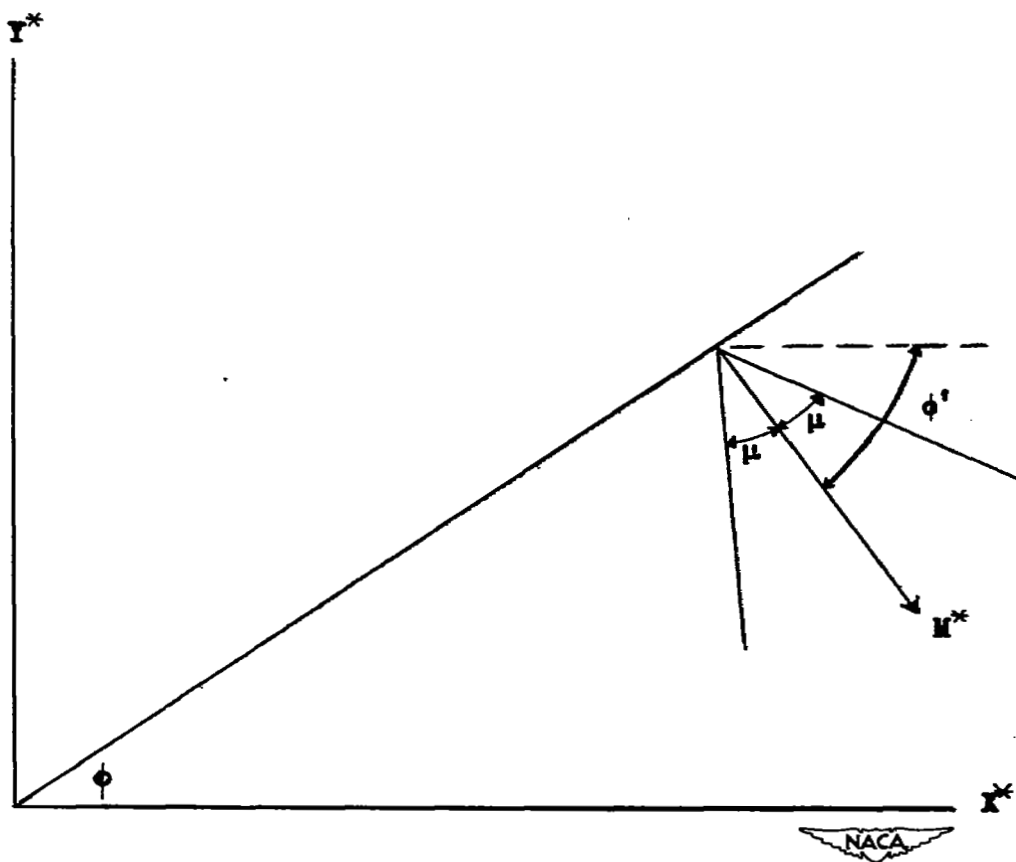


Figure 2.- Geometry of vortex field.



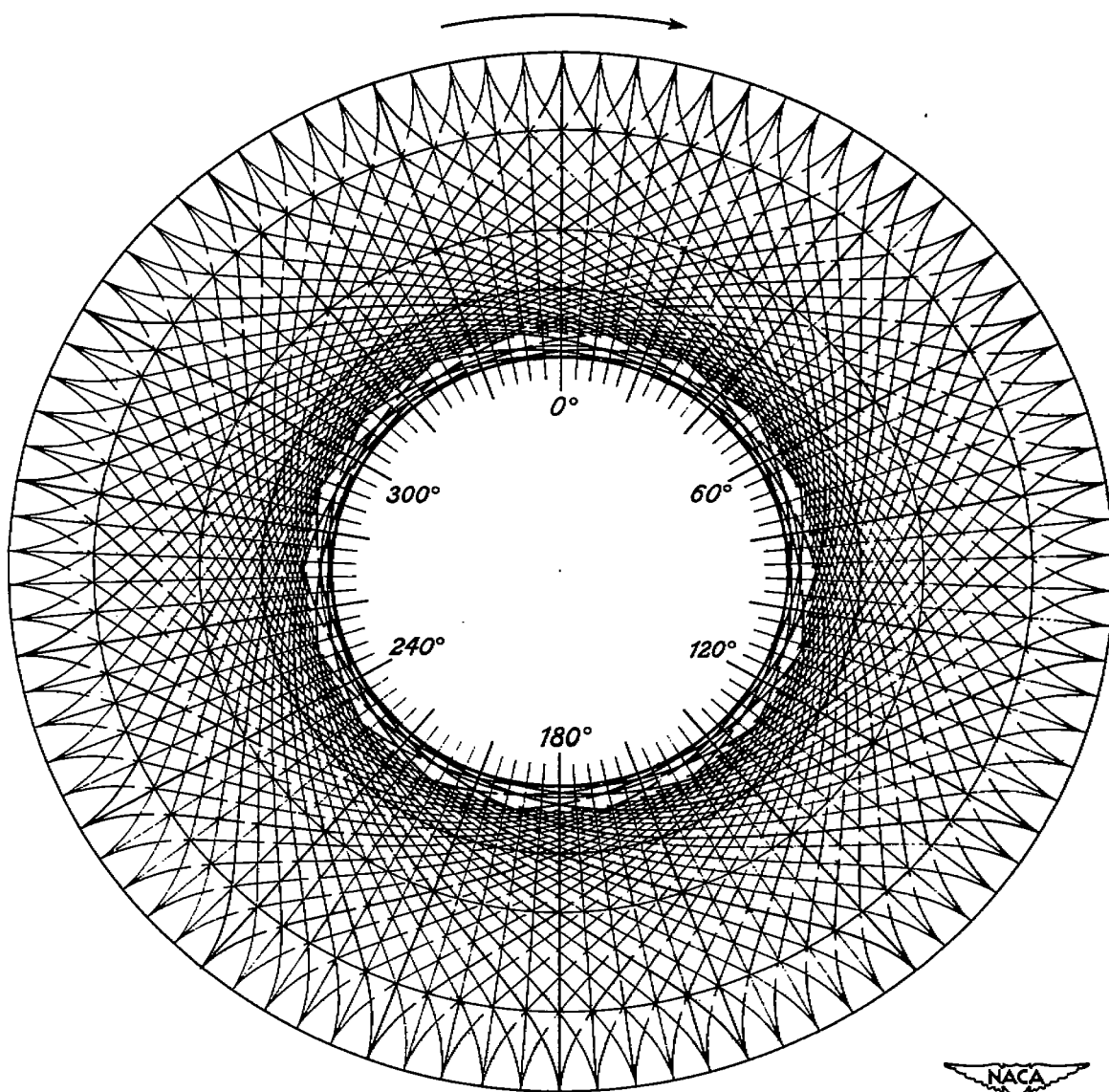


Figure 3.- Characteristic line network for supersonic vortex flow according to A. Busemann.  $\gamma = 1.40$ .

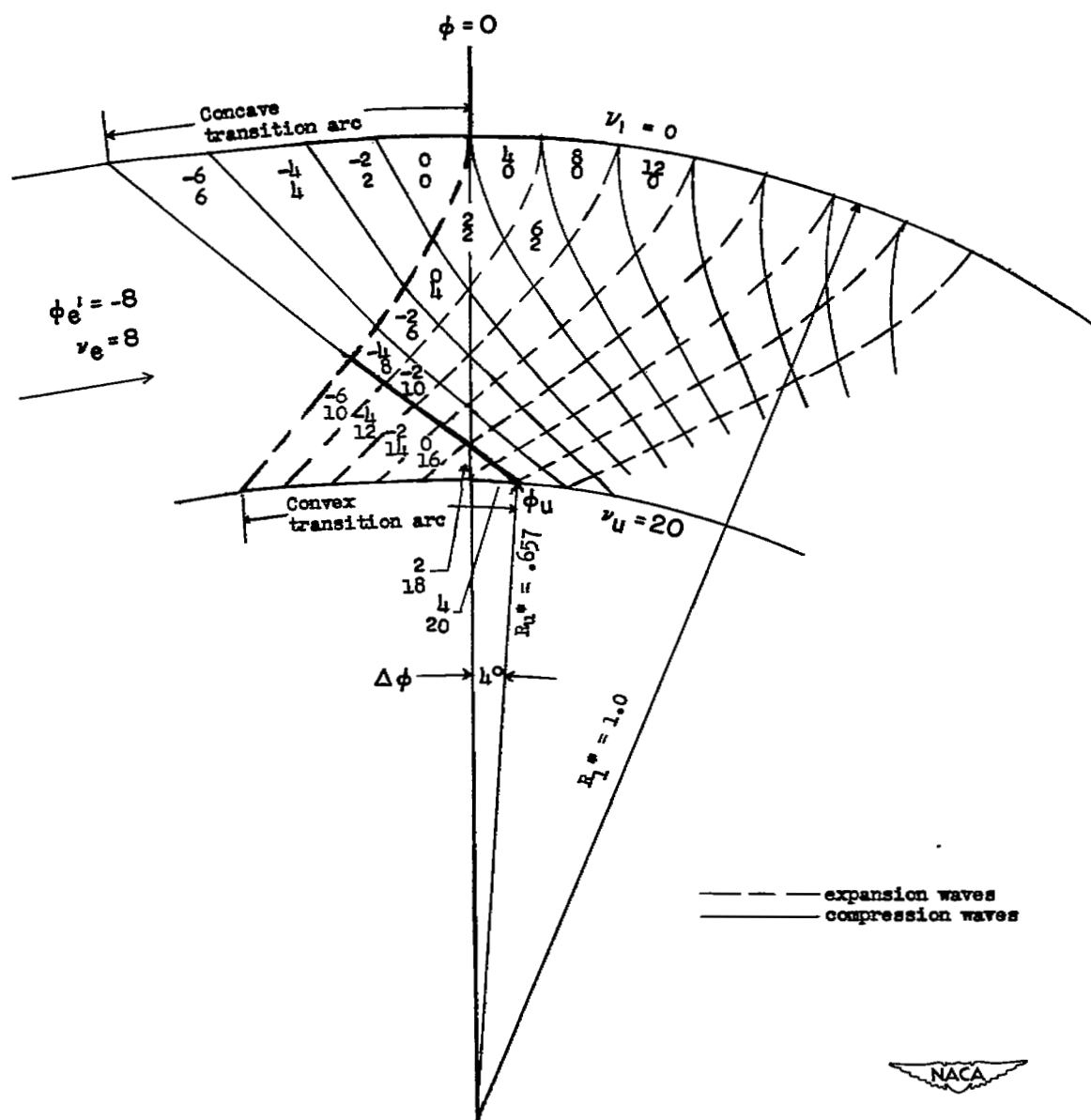


Figure 4.- Construction of transition arcs.

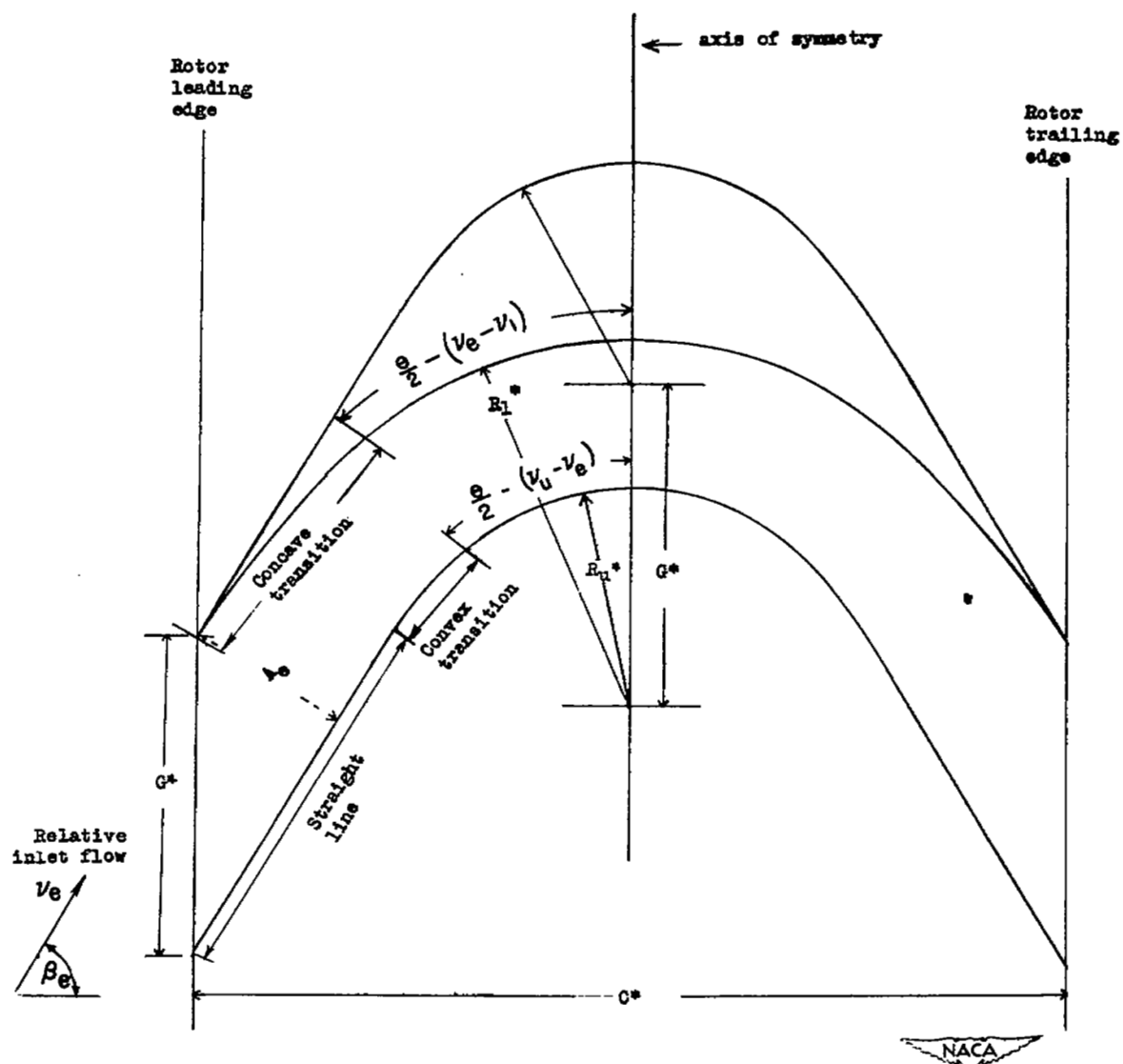


Figure 5.- Construction of typical symmetrical blade section.

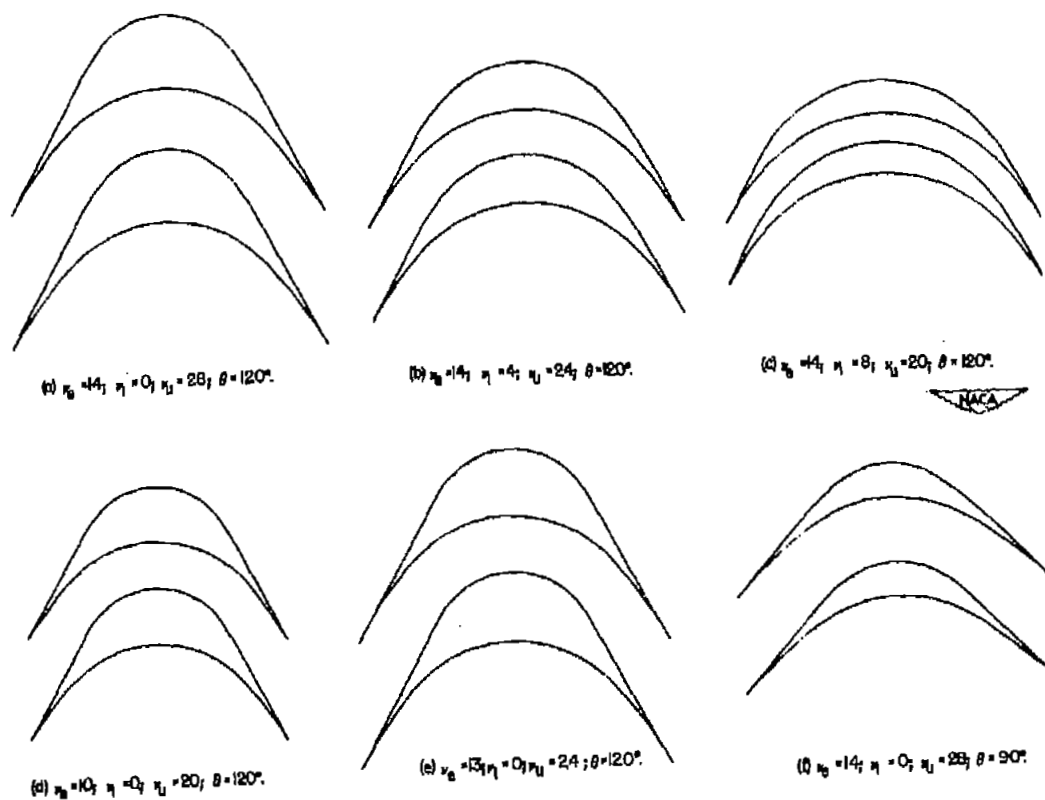


Figure 6.- Examples of blades.

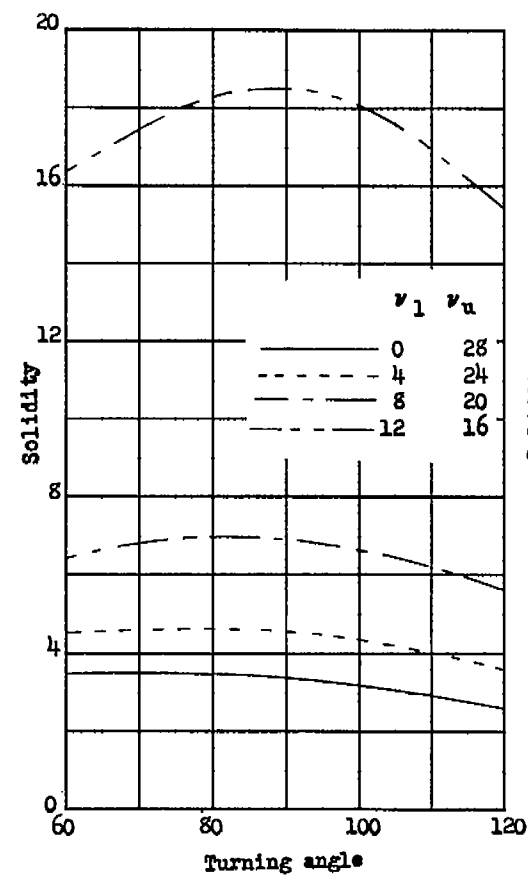
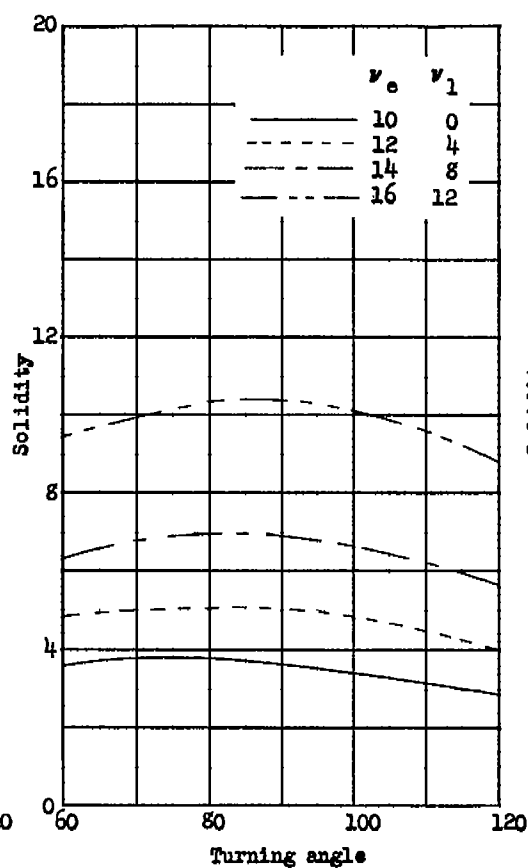
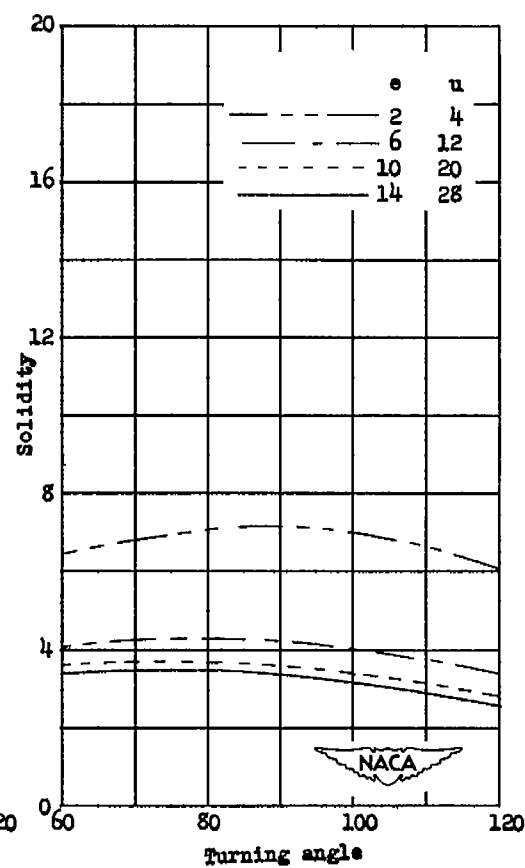
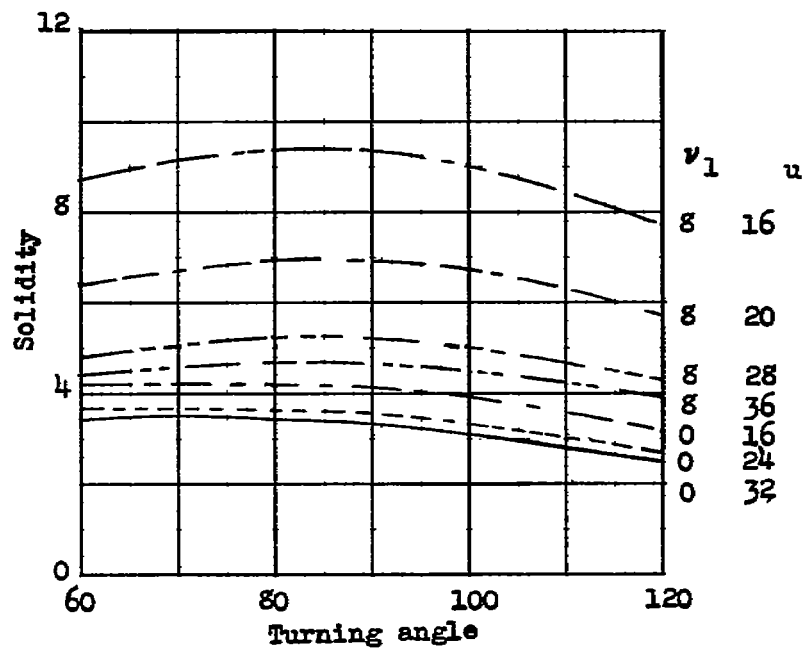
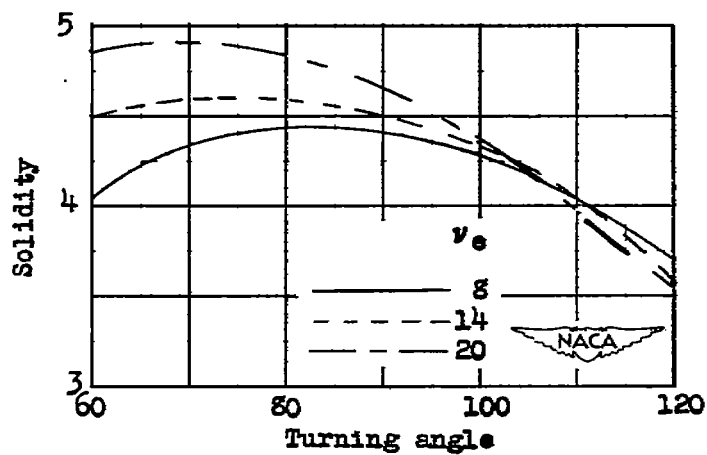
(a)  $v_e = 14$ .(b)  $v_u = 20$ .(c)  $v_1 = 0$ .

Figure 7.- Variation of solidity with turning angle for special case where  

$$v_e = \frac{1}{2}(v_u + v_1).$$

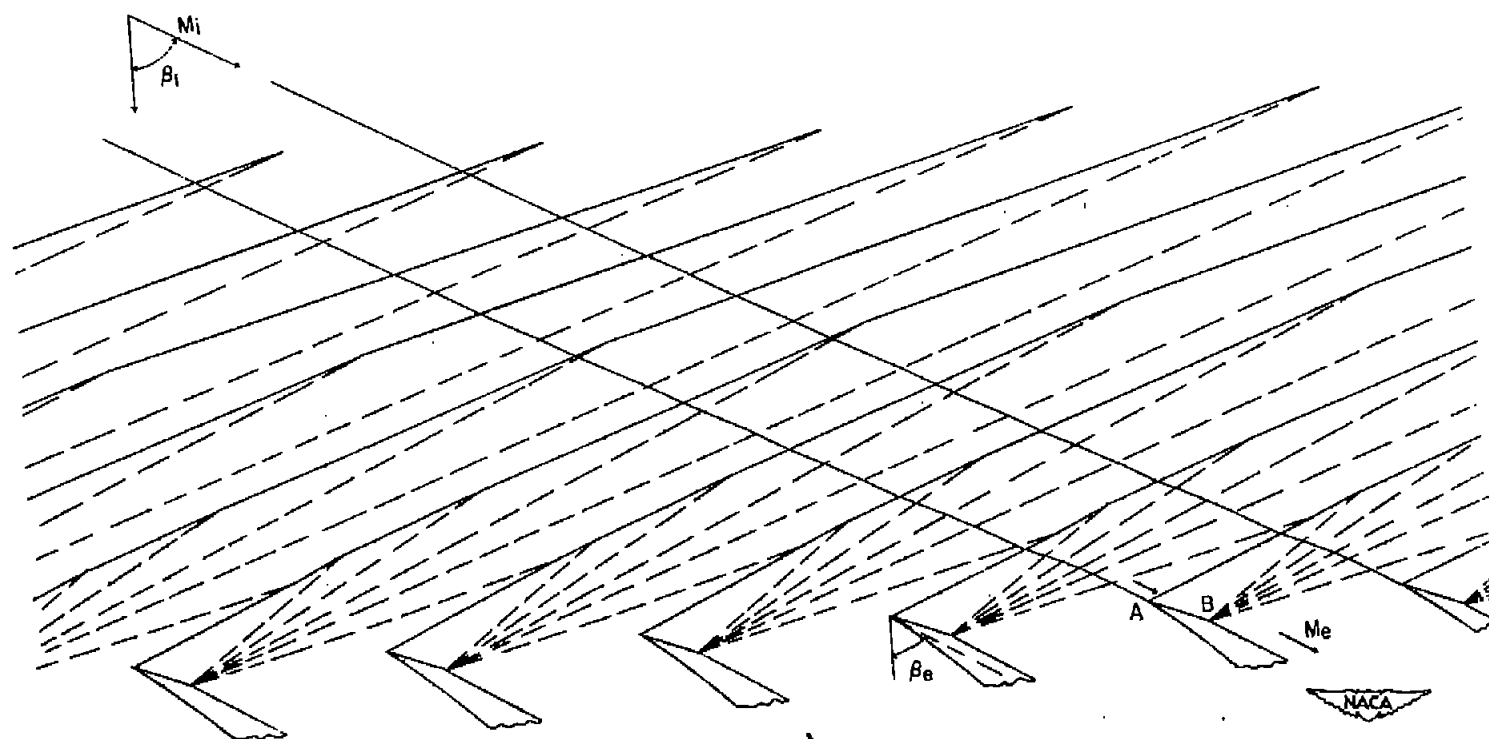


(a)  $v_e = 14$ .



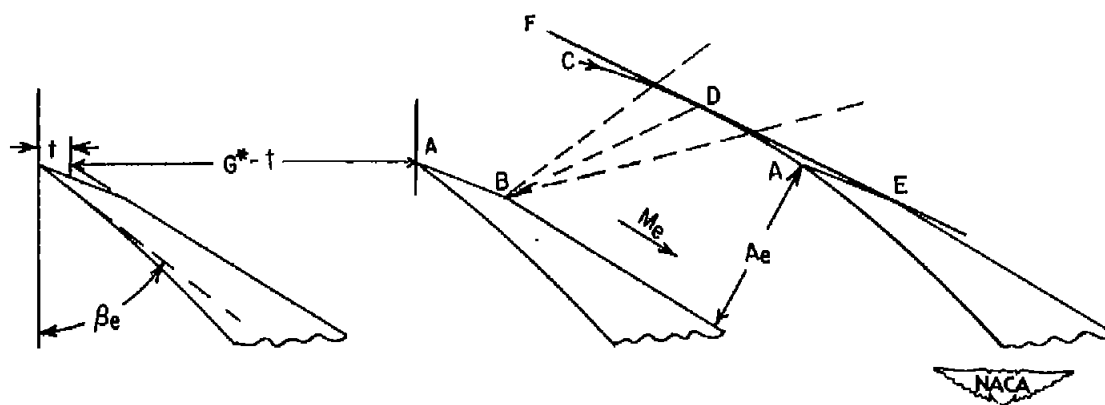
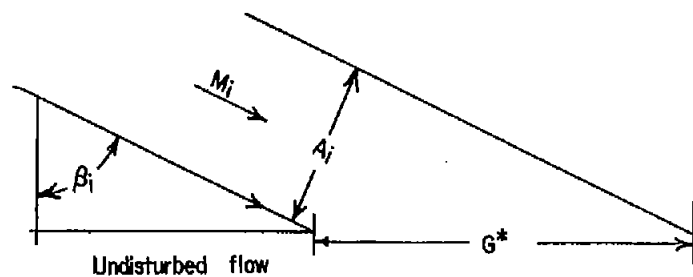
(b)  $v_u = 24$  and  $v_l = 4$ .

Figure 8.- Variation of solidity with turning angle for general case.



(a) Subsonic axial velocity component, external waves.

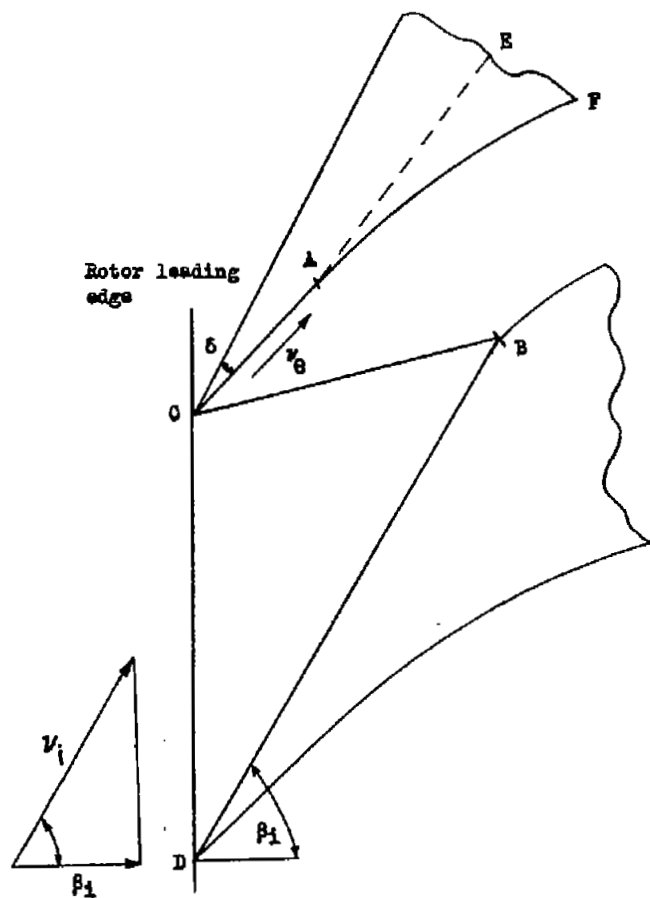
Figure 9.- Methods of increasing section thickness.



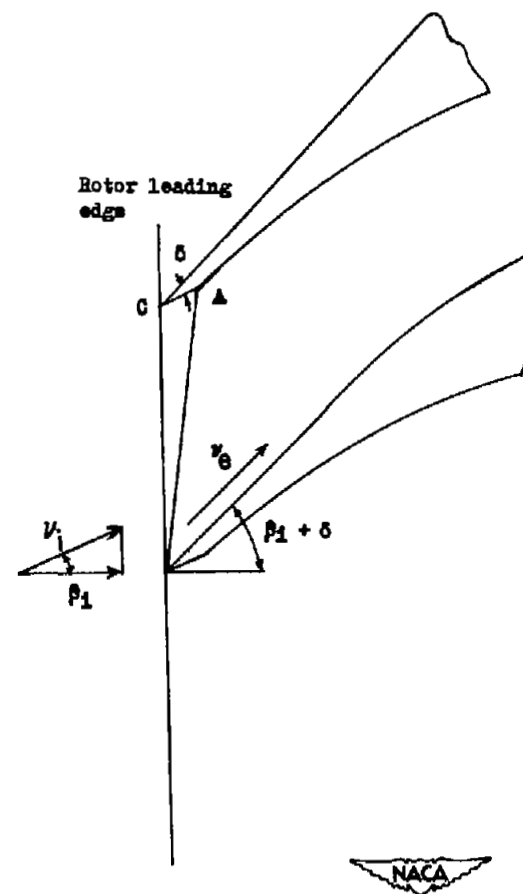
(b) Flow geometry for external waves.

Figure 9.- Continued.





(c) Subsonic axial-velocity-component shock originating on concave surface.



(d) Supersonic axial-velocity component.

Figure 9.- Concluded.

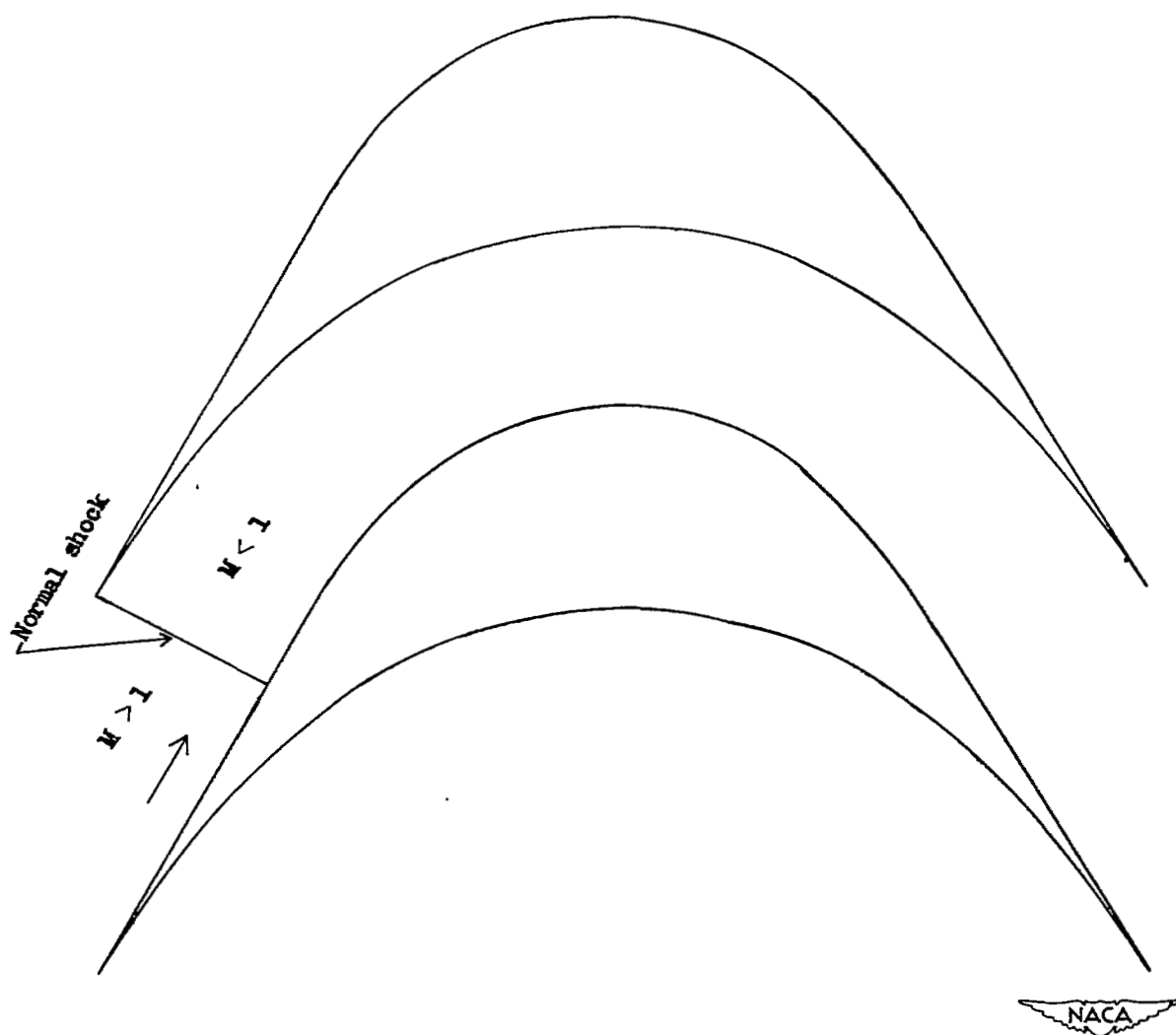


Figure 10.- Diagram of flow in the channel at the instant immediately before starting.

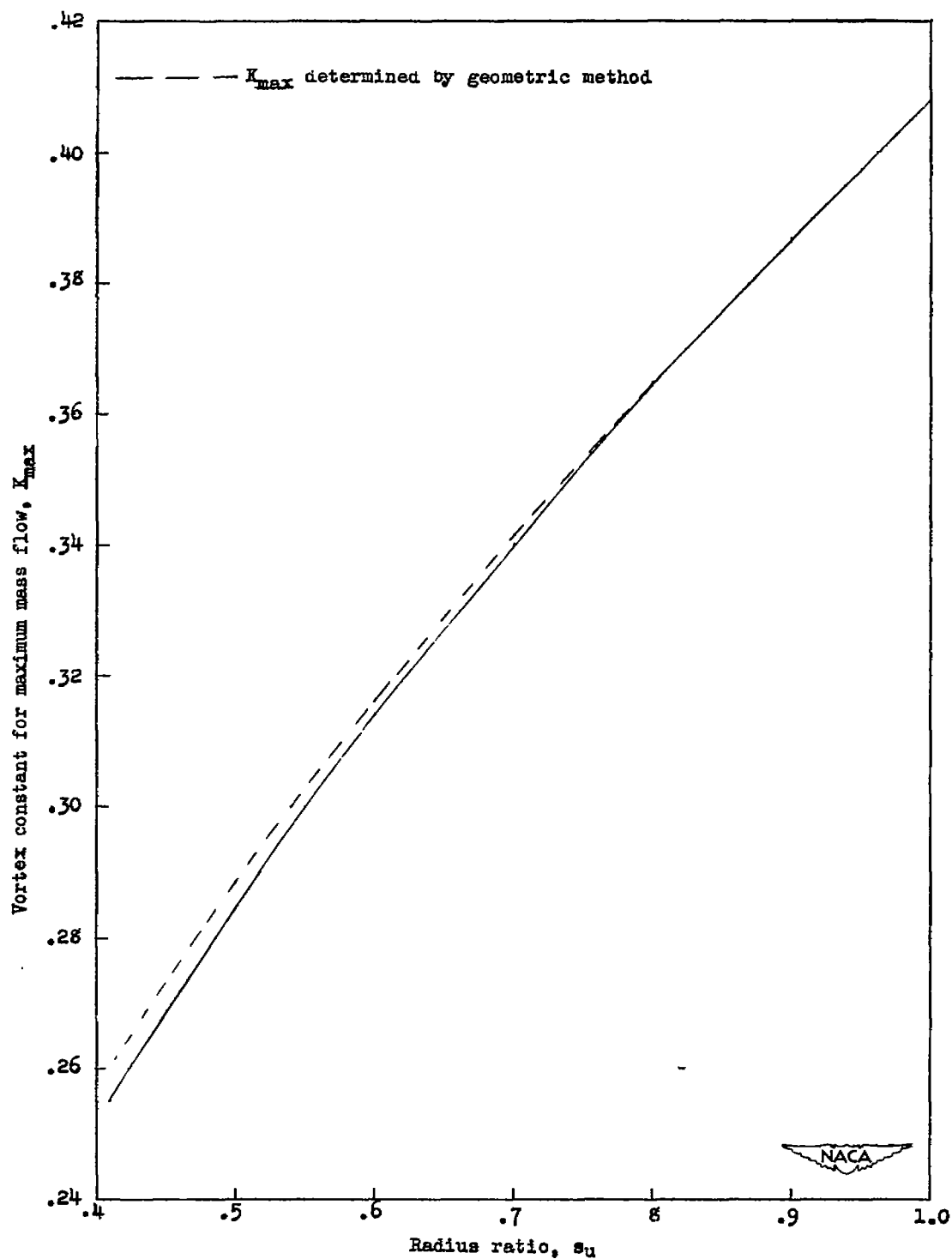


Figure 11.- Value of  $K_{\max}$  as function of radius ratio for  $\gamma = 1.40$ .

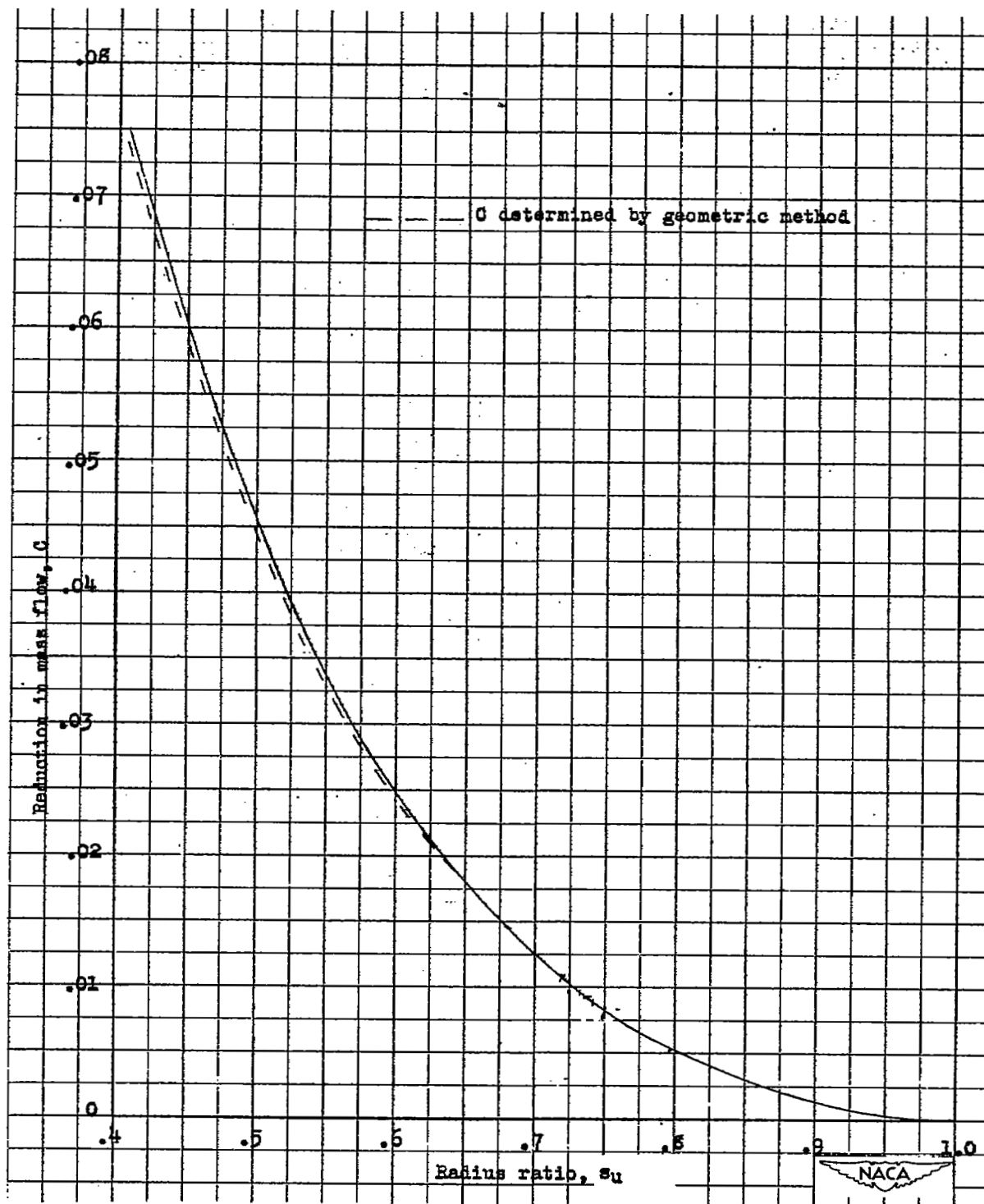


Figure 12.- Reduction in maximum mass flow as function of radius ratio for  $\gamma = 1.40$ .

Note:  $s_u = \frac{R_u^*}{R_c^*} = \frac{M_c^*}{M_u^*}$

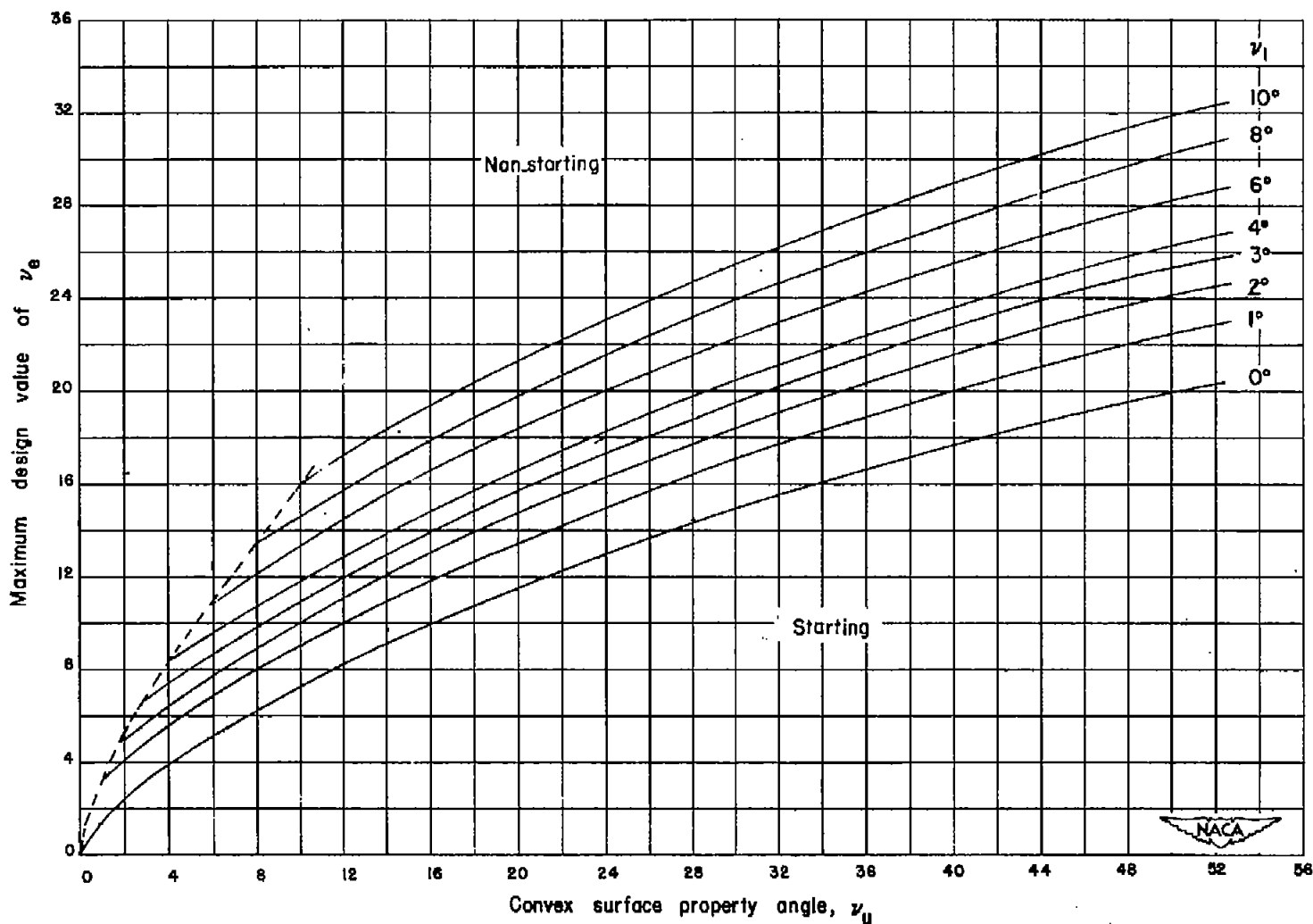


Figure 13.- Maximum design-inlet value of  $v_e$  for starting as a function of  $v_1$  and  $v_u$ .

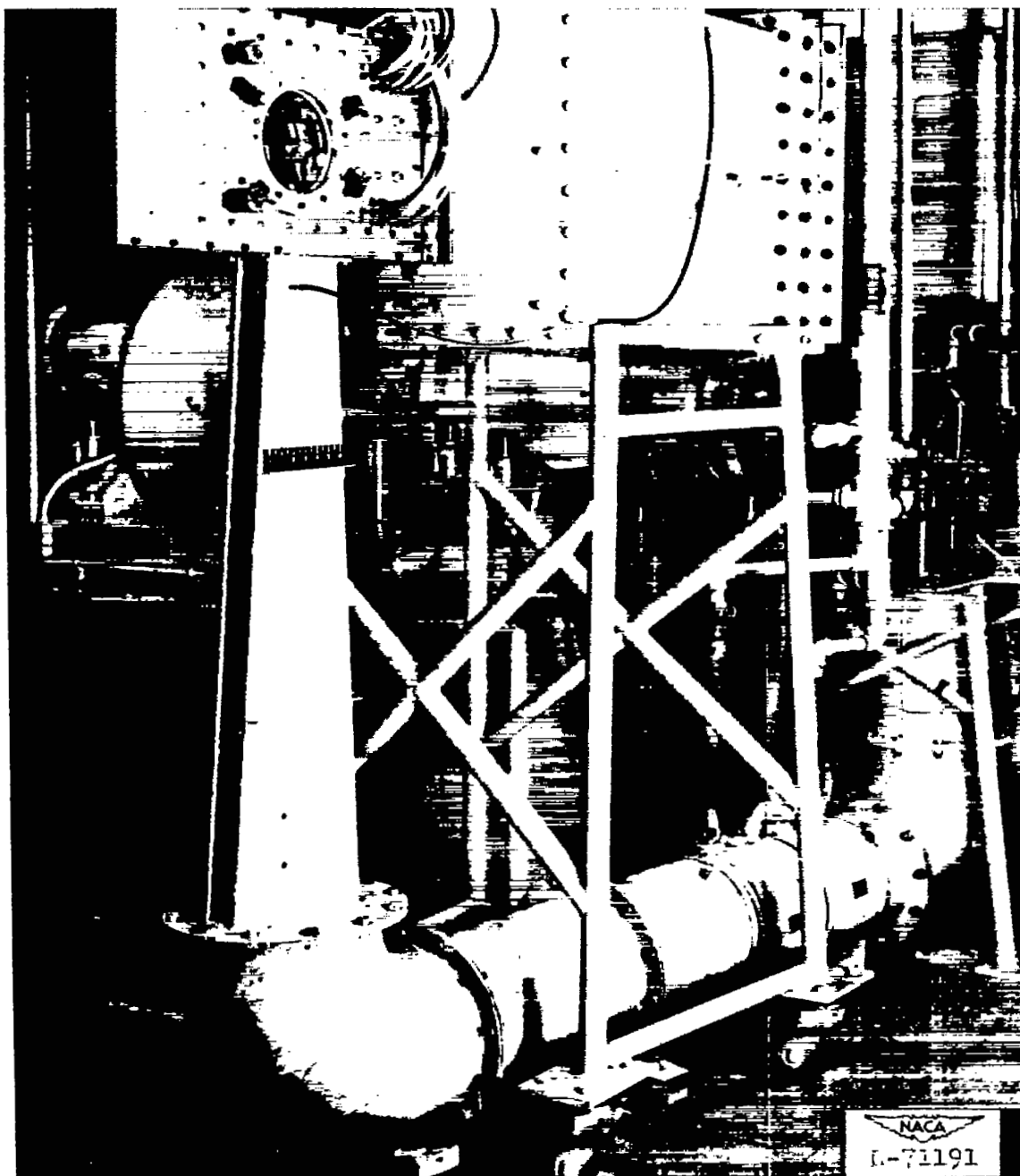


Figure 14.- Photo of  $2\frac{1}{4}$ -by 2-inch supersonic cascade tunnel.

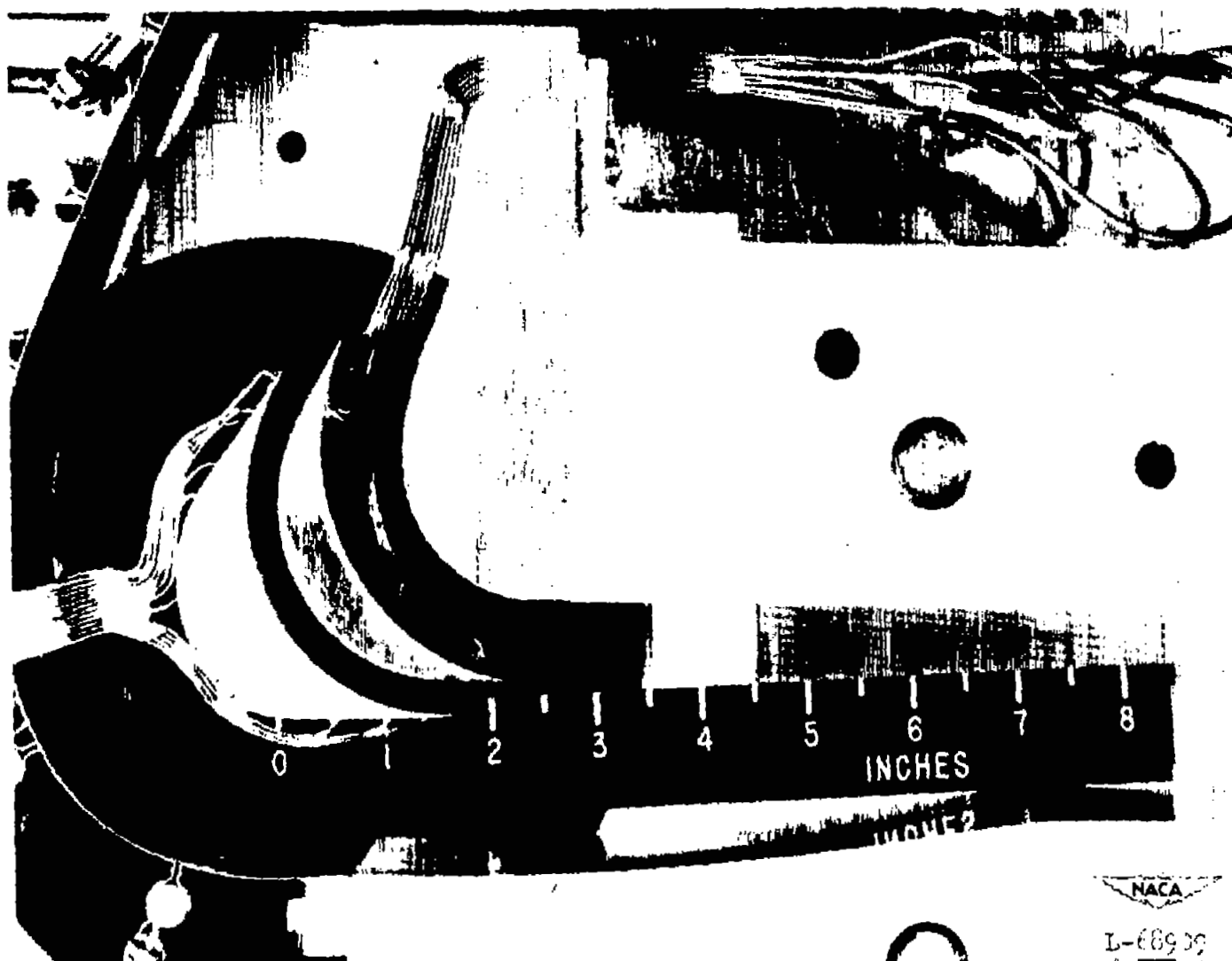


Figure 15.- View of test section with one side wall removed.

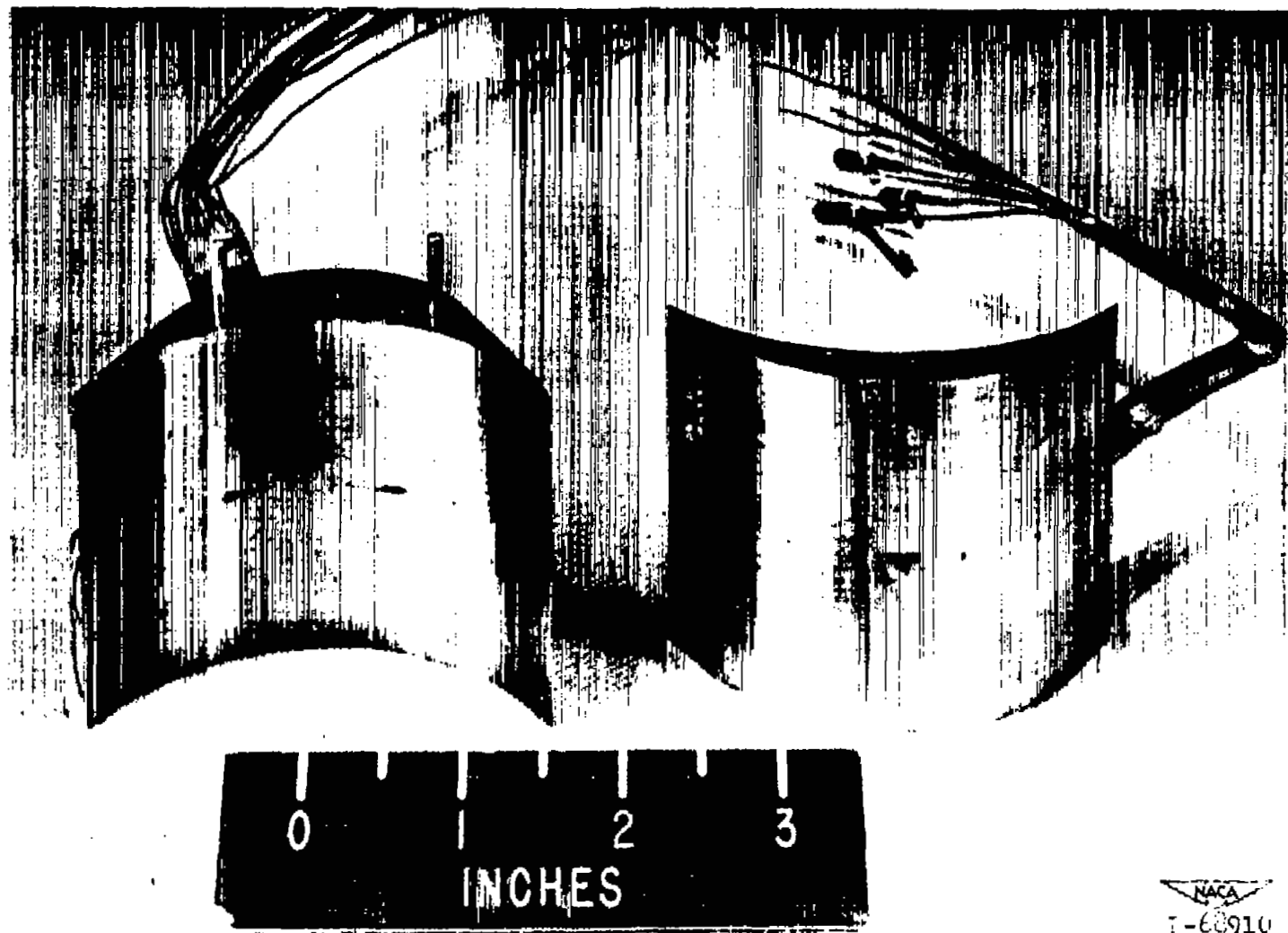


Figure 16.- Close up of blade I showing the static-pressure orifices and scratches.



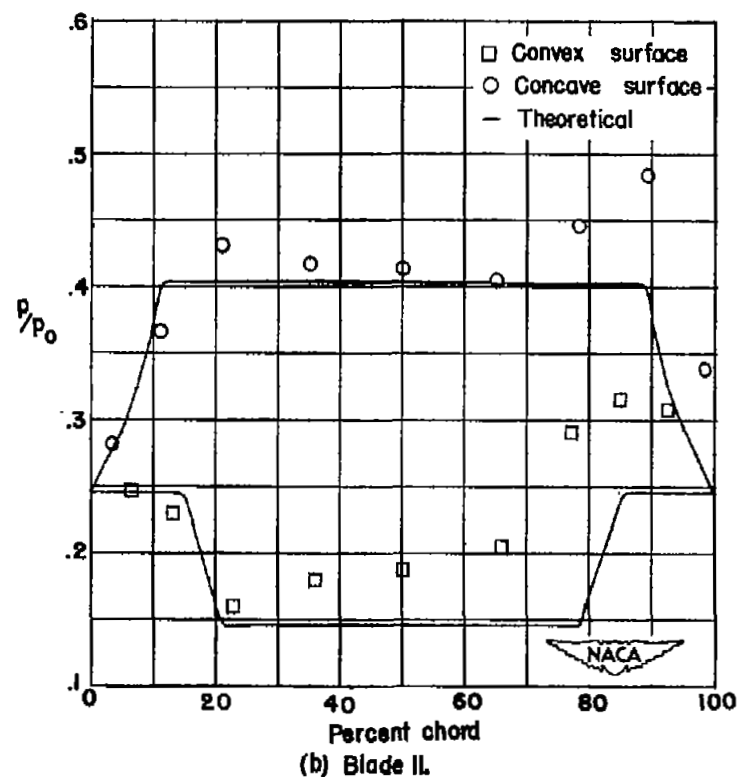
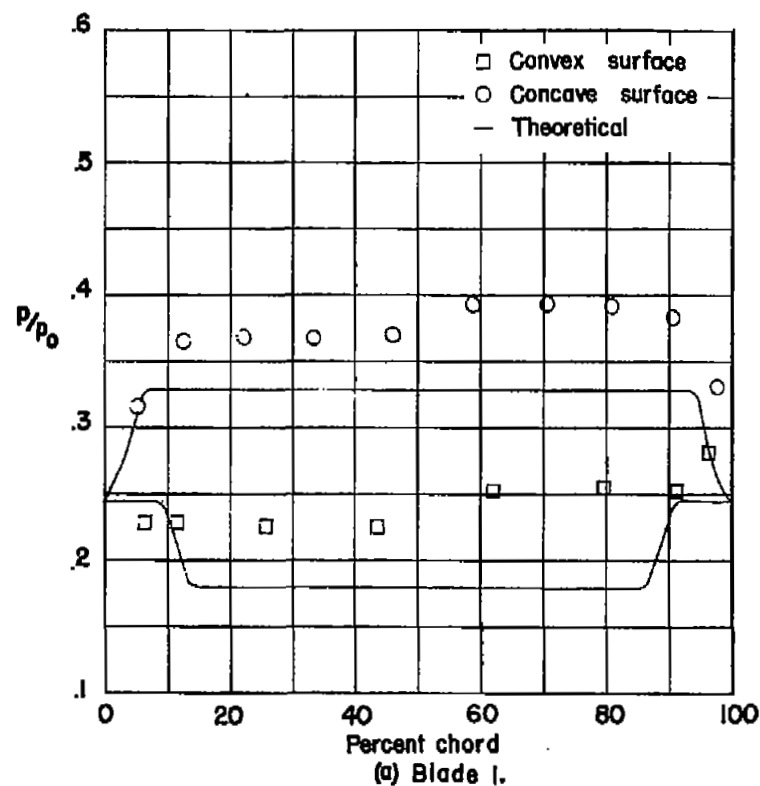


Figure 17.- Pressure distribution along concave and convex surfaces.

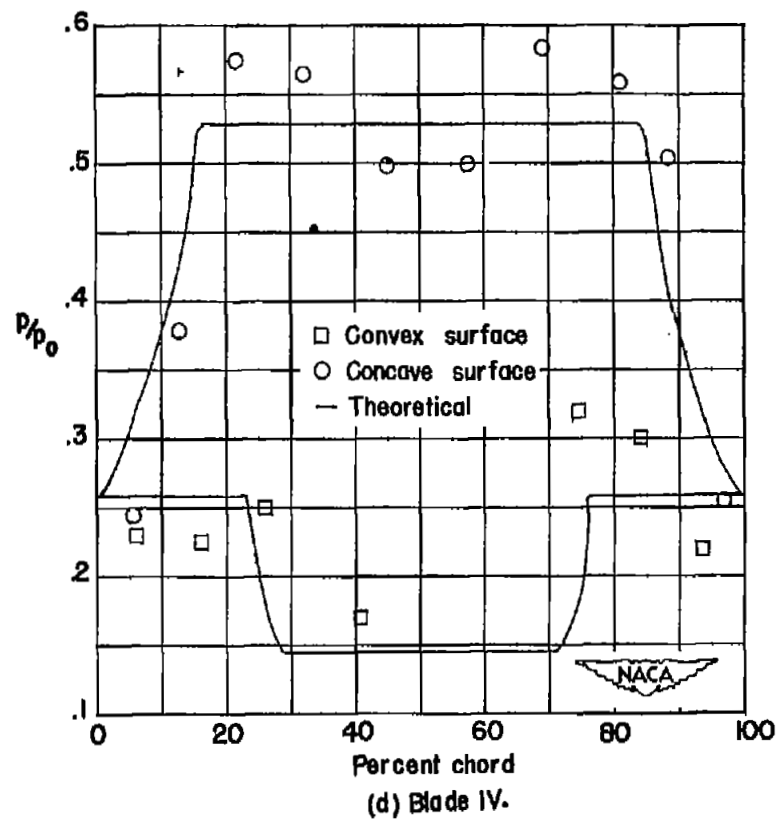
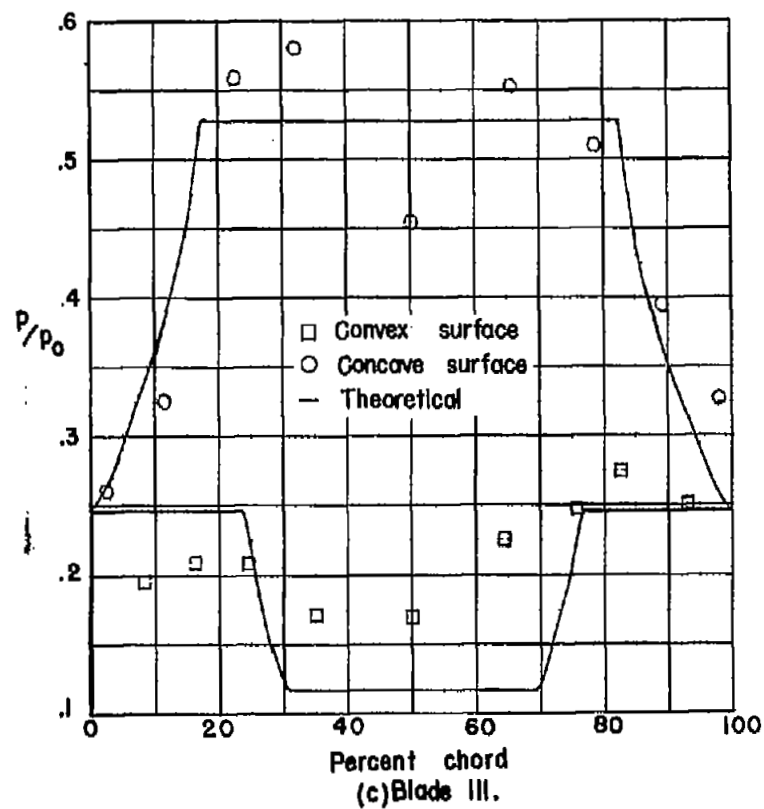


Figure 17.- Concluded.

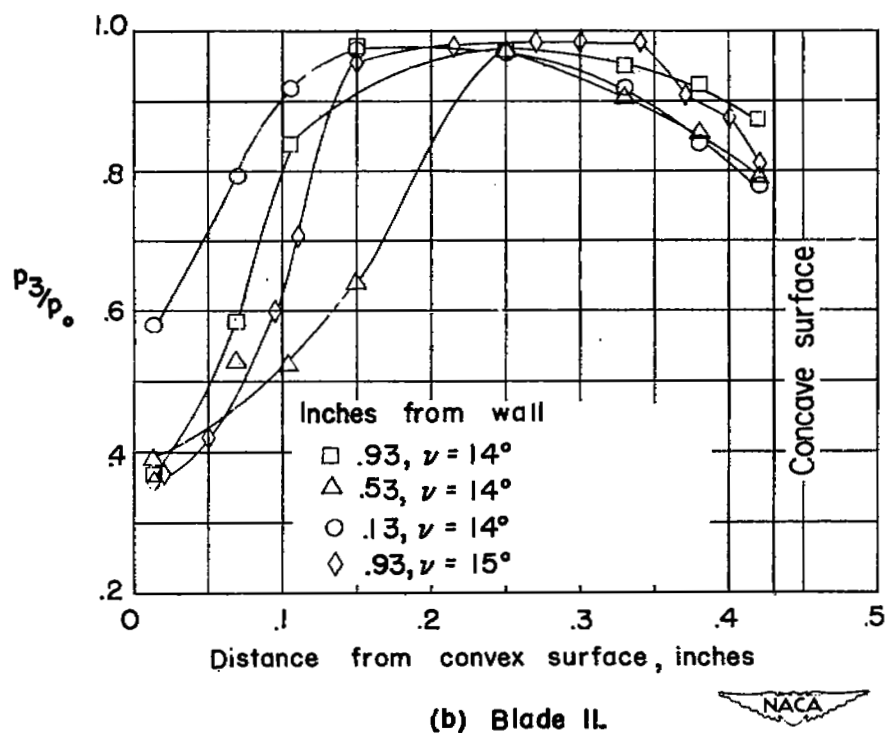
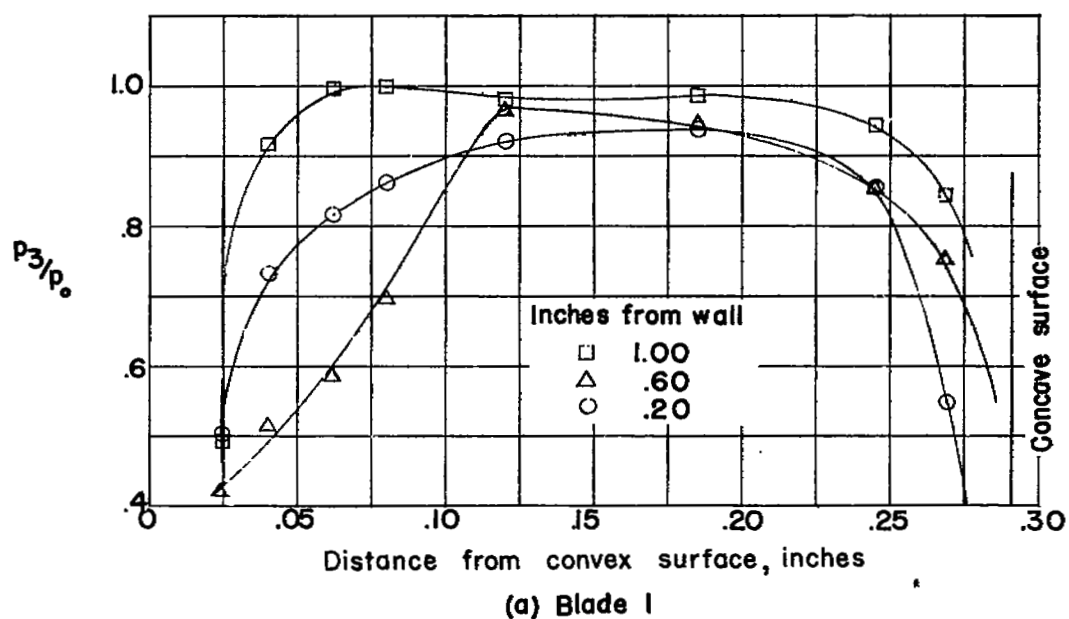


Figure 18.- The variation of the stagnation-pressure recovery near end of passage.

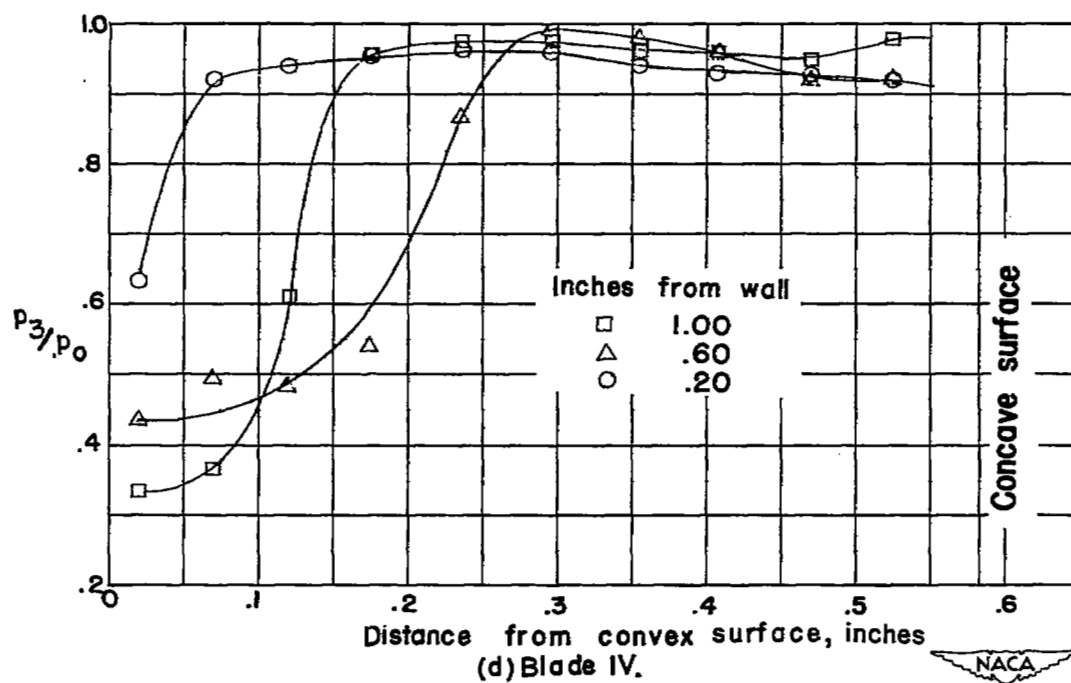
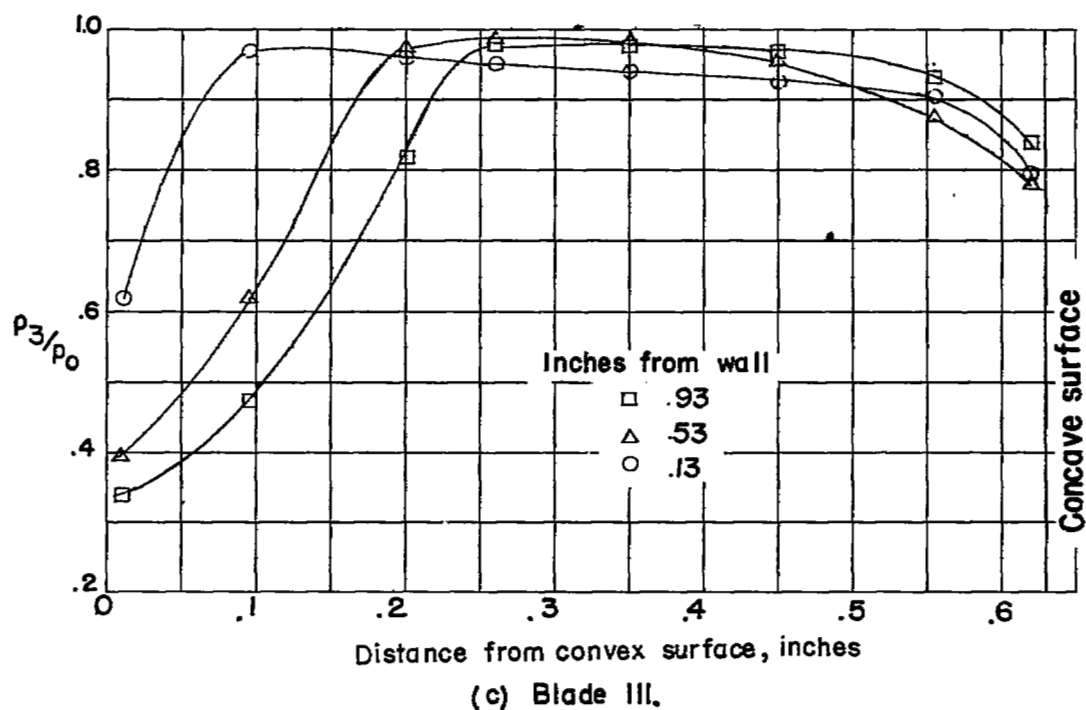
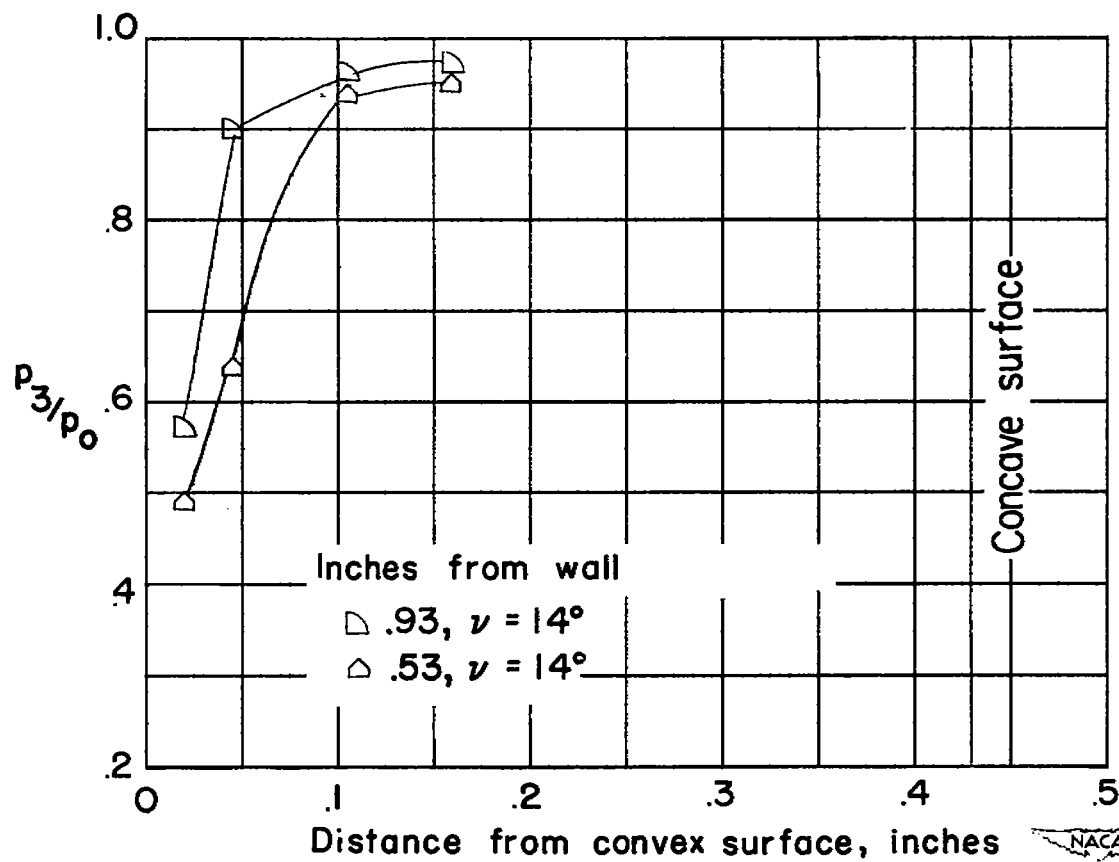


Figure 18.- Continued.



(e) Blade II, with fence.

Figure 18.- Concluded.



(a) Total-pressure tube in place.

L-72736

Figure 19.- Schlieren photographs of the flow in the passage of blade I.  
 $v_e = 15^\circ$ .



(b) Total-pressure tube removed.



Figure 19.- Continued.

L-72737



L-72738

(c) Flexible wall moved closer to trailing edge of blades.

Figure 19.- Concluded.





(a)  $\alpha_e = 14^\circ$ .



L-72739

Figure 20.- Schlieren photographs of the flow in the passage of blade II.

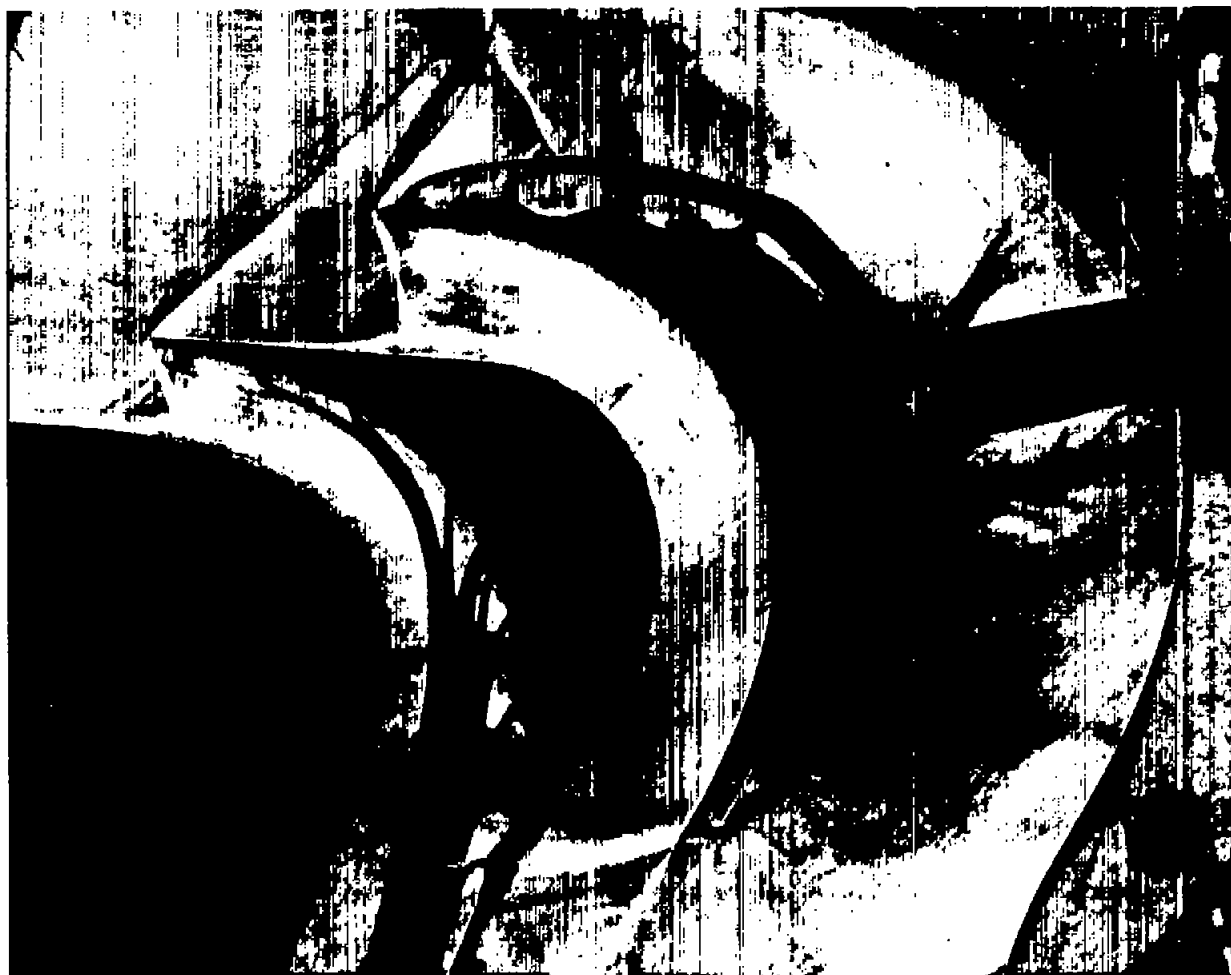


(b)  $v_e = 15^\circ$ .

Figure 20.- Concluded.



L-72740



(a) Passage not started.  $v_e = 14^\circ$ .

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L-72741

Figure 21.- Schlieren photographs of the flow in the passage of blade III.

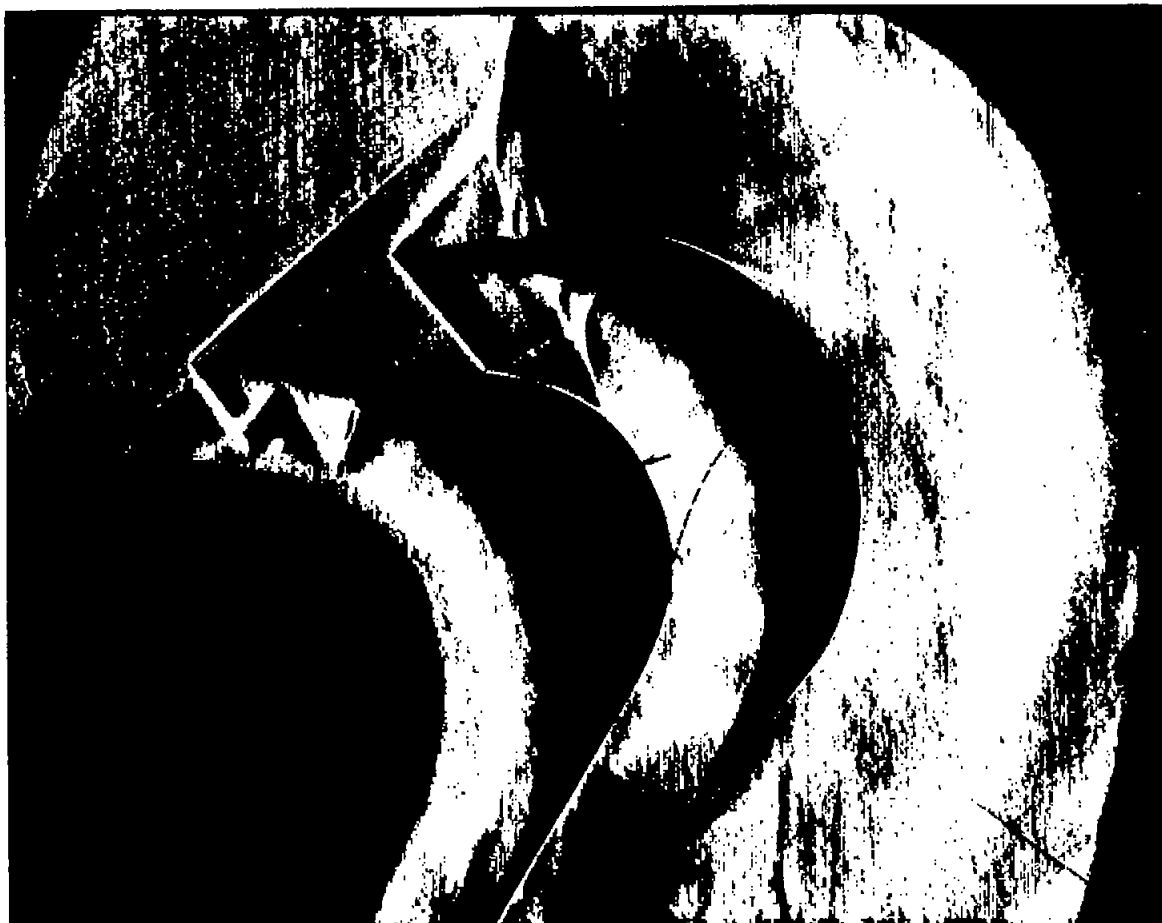


(b) Passage started.  $v_e = 18^\circ$ .



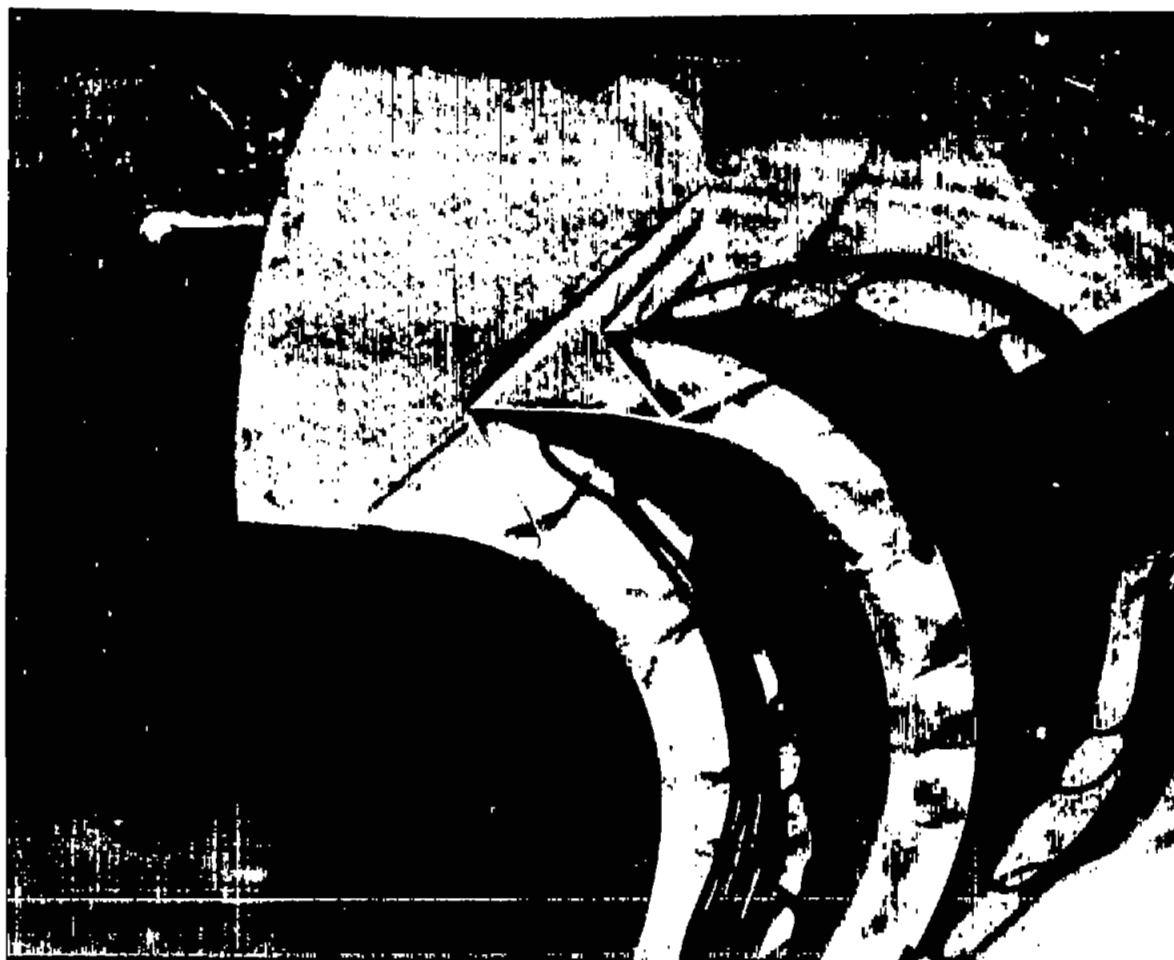
Figure 21.- Concluded.

L-72742



L-72743

Figure 22.- Schlieren photographs of the flow in the passage of blade IV.  
 $v_e = 15^\circ$ .



L-72744

Figure 23.- Schlieren photograph of the flow for an incorrectly spaced pair of blades.

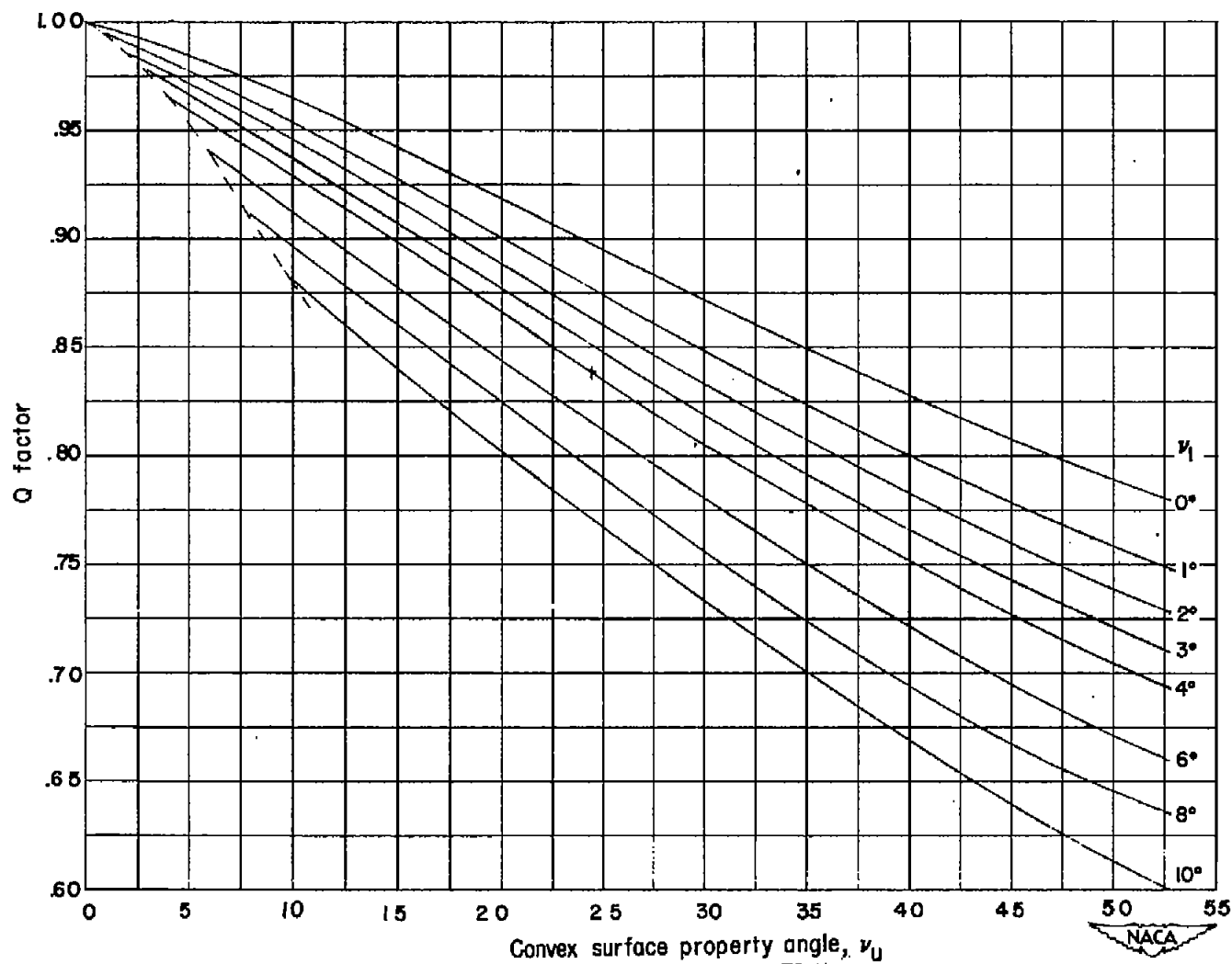
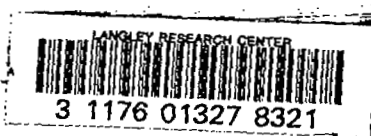


Figure 24.- Q factor for solution of maximum inlet Mach number as a function of  $v_u$  and  $v_1$ .

# SECURITY INFORMATION

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